

Structural Durability of MTH Steel

Mubea

 Fraunhofer
LBF

Mubea TRB – USA

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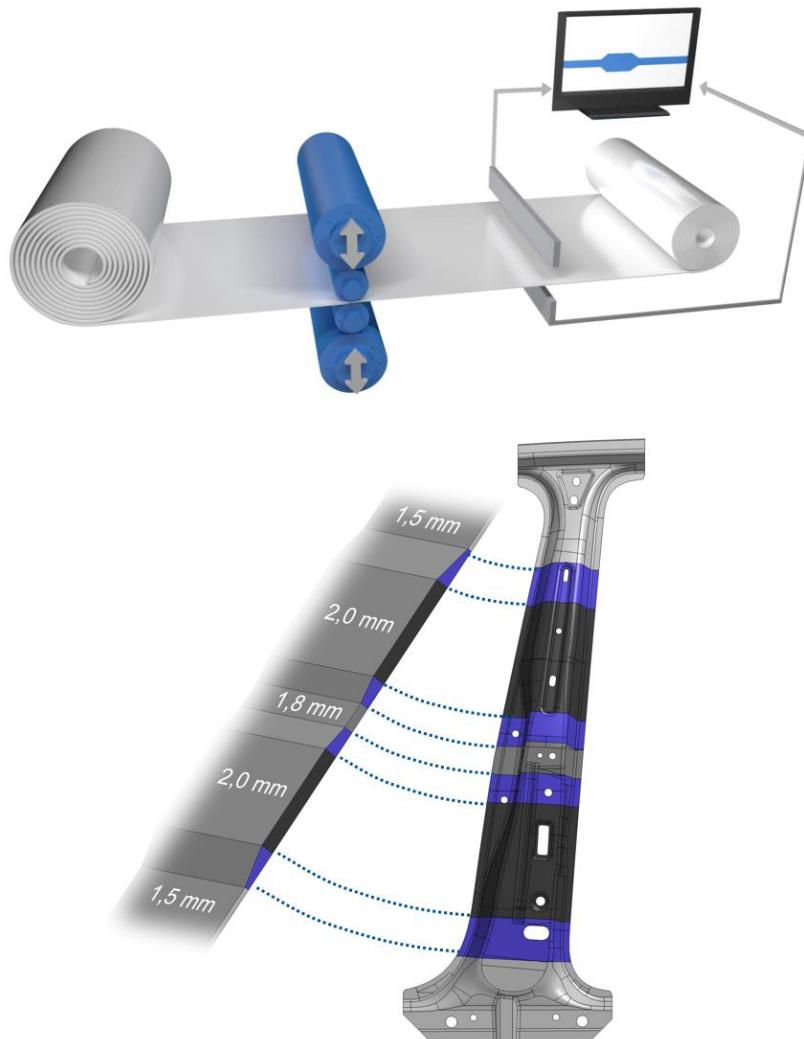
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Tim Korschinsky

Dr.-Ing. Rainer Wagener

GREAT DESIGNS IN
STEELTM



Concept

- › Cost efficient lightweight parts with load and function-optimized material usage

Implementation

- › Flexible Cold-Rolling Process
- › Flat material with repeated, varying thickness runs and harmonious transition zones
- › Thickness run optimization drives the cost efficiency

Targets / Benefits

- › Functional Improvement (Performance, Manufacturing, Quality)
- › Part Integration / Design Simplification
- › Weight Reduction / Cost Reduction

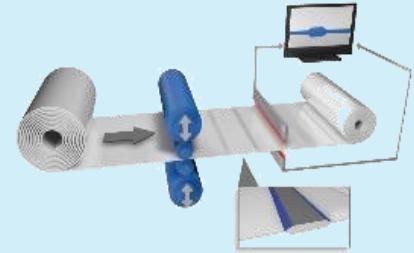
Mat. Innovation – Mubea TailorHardened (MTH)

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Conventional → TRB cold-forming process flow



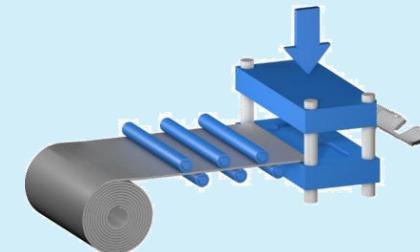
Raw Material
(HR500LA)



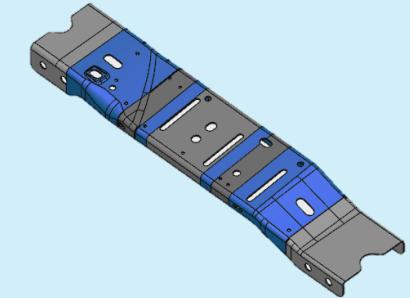
Flex Rolling



Batch Annealing



Part Production

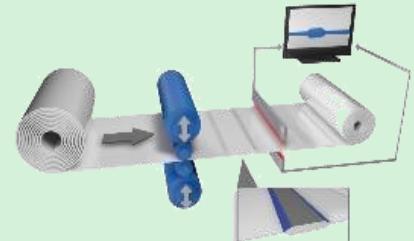


Homogeneous
mechanical properties
"HC380 LA TRB"

MTH → cost effective process flow



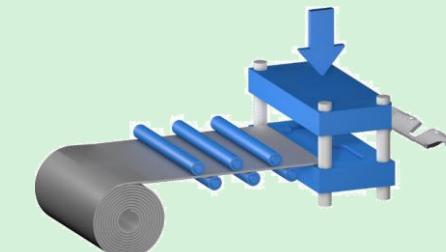
Raw Material
(DD13)



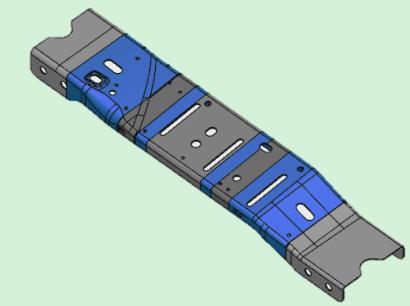
Flex Rolling



Batch Annealing



Part Production



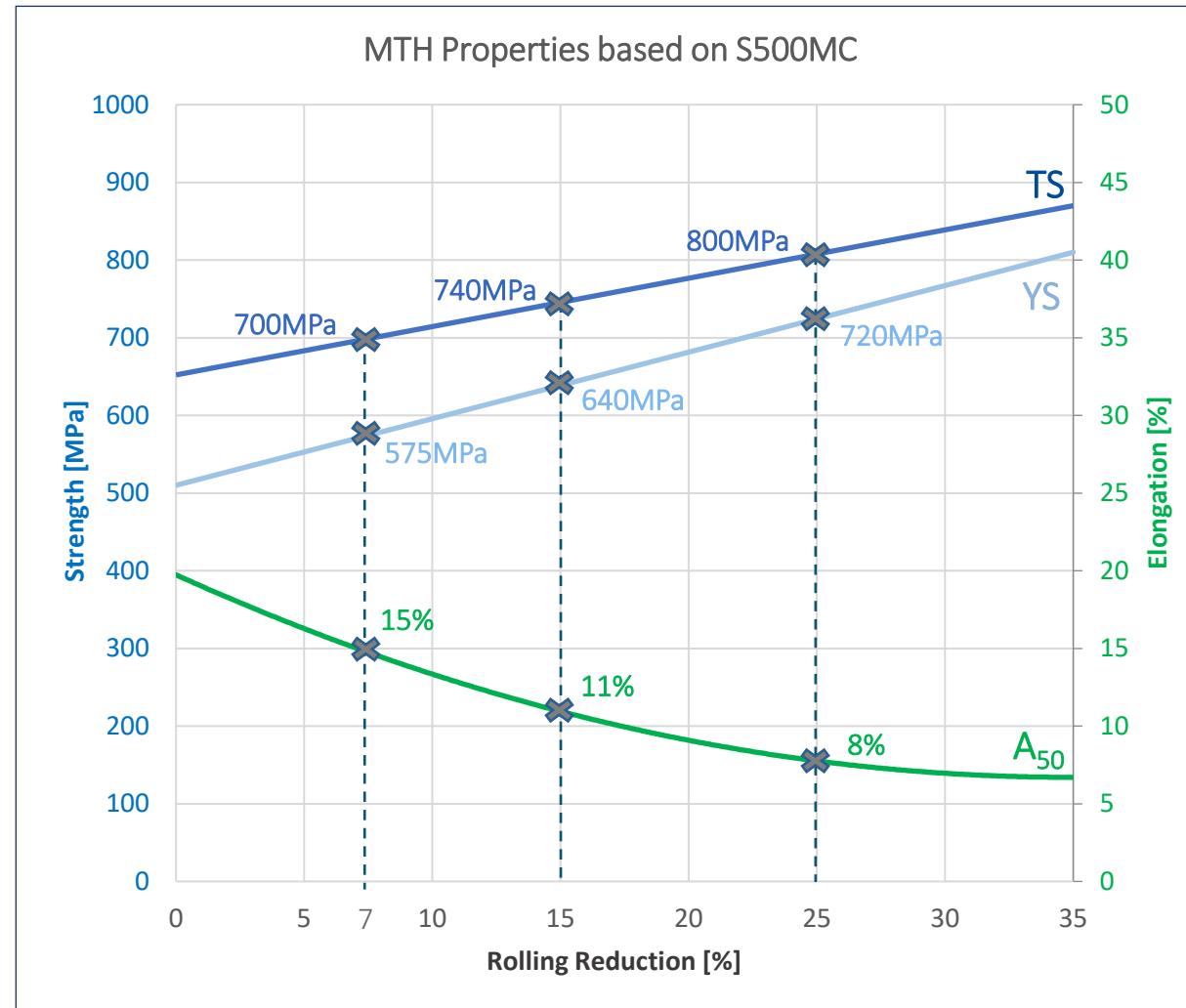
TailorHardened
mechanical properties
"MTH 260Y/530Y TRB"

Manufacturing cost reduction through shorter process flow and less expensive raw material!

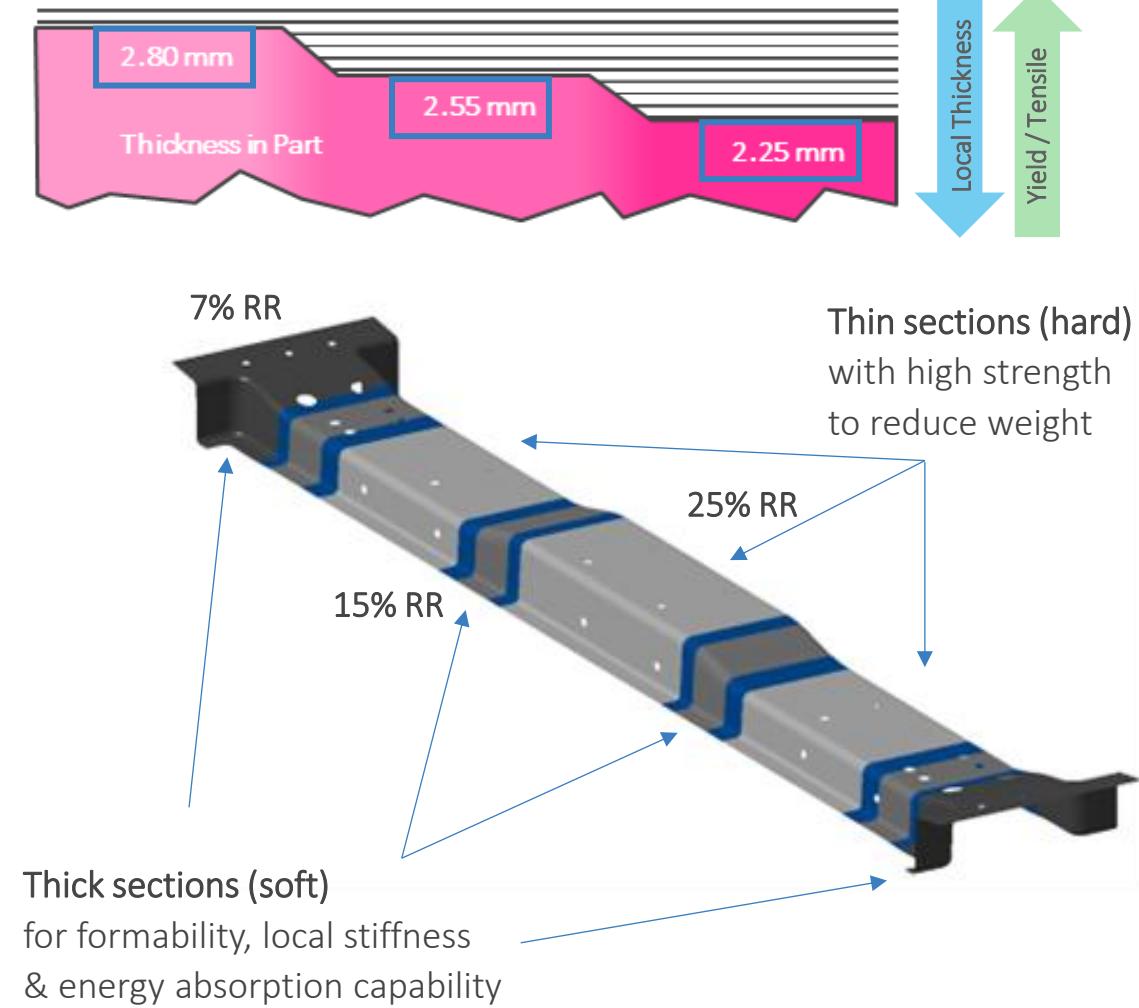
MTH – Mechanical Properties Behavior

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Example: MTH575Y/720Y from HR500LA



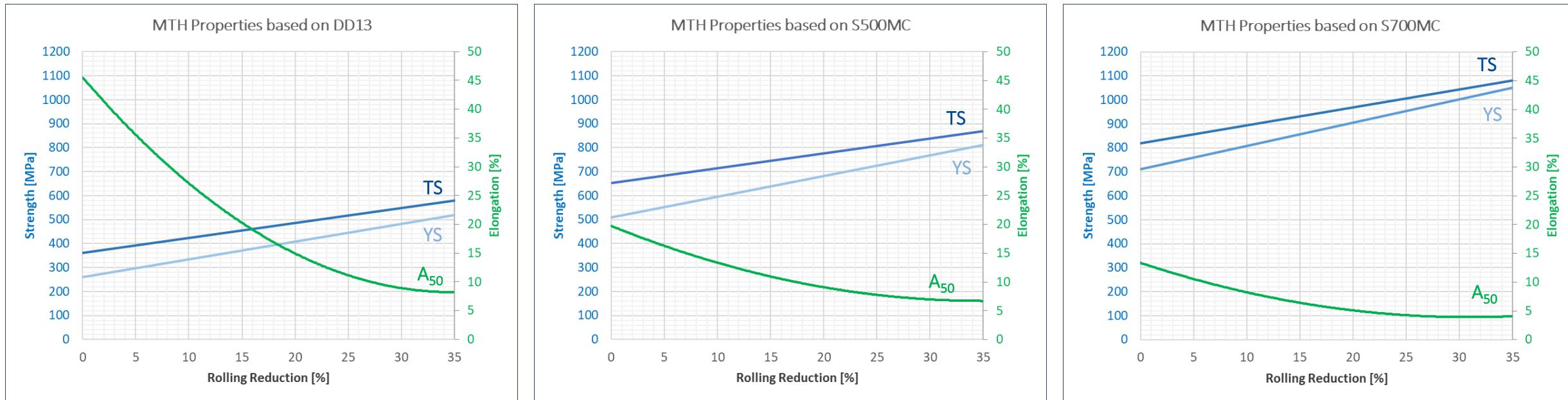
Raw Material Gauge: 3.0 mm



Cost Efficient Component Design & Manufacturing with Local Tailored Properties!

MTH – Mat. Characterization Status & Next Steps

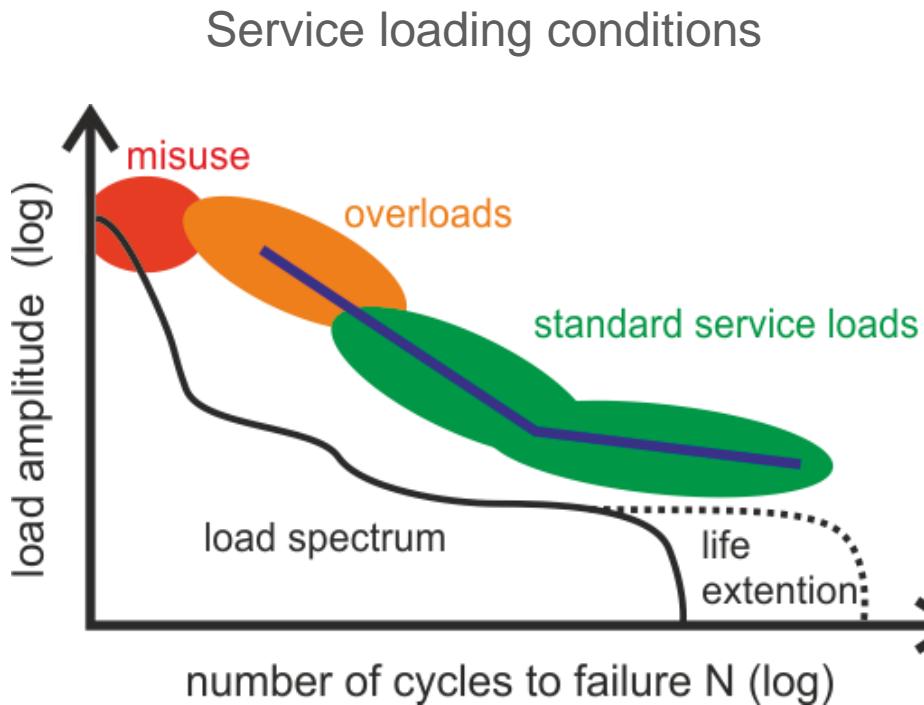
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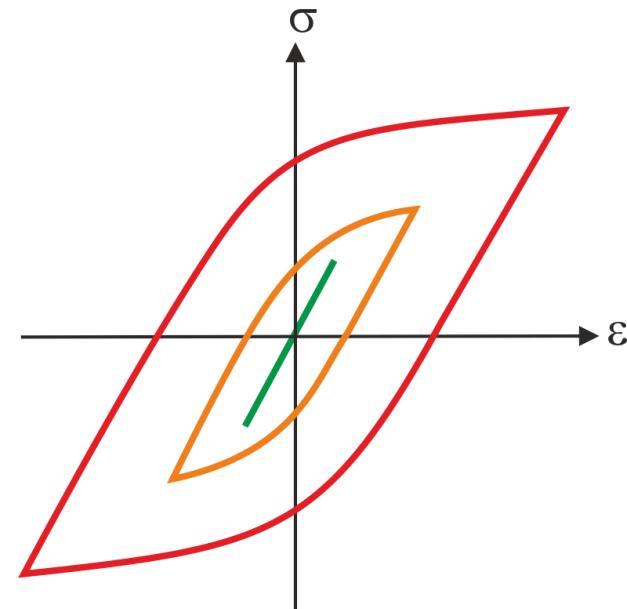
- › MTH → Flex Cold-Rolled Steel based on HSLA grades → increasing Yield and Tensile Strength through Dislocation Hardening
- › **High gauge** → mechanical properties slightly elevated due to minor Rolling Reduction with **Higher Elongation Capability**
- › **Low gauge** → enhanced mechanical properties due to major Rolling Reduction with **Higher Yield & Tensile Strength Levels**
- › **How about Fatigue** → Do MTH steels show comparable increased Fatigue Strength? How to predict it reliably?

Next Steps = Fatigue Assessment Plan → Specimen Tests + Math Approach + Part Validation

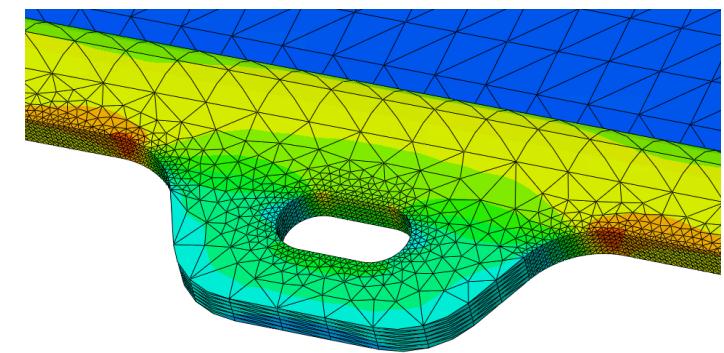
Motivation – Fatigue Approach



Stress-strain behavior

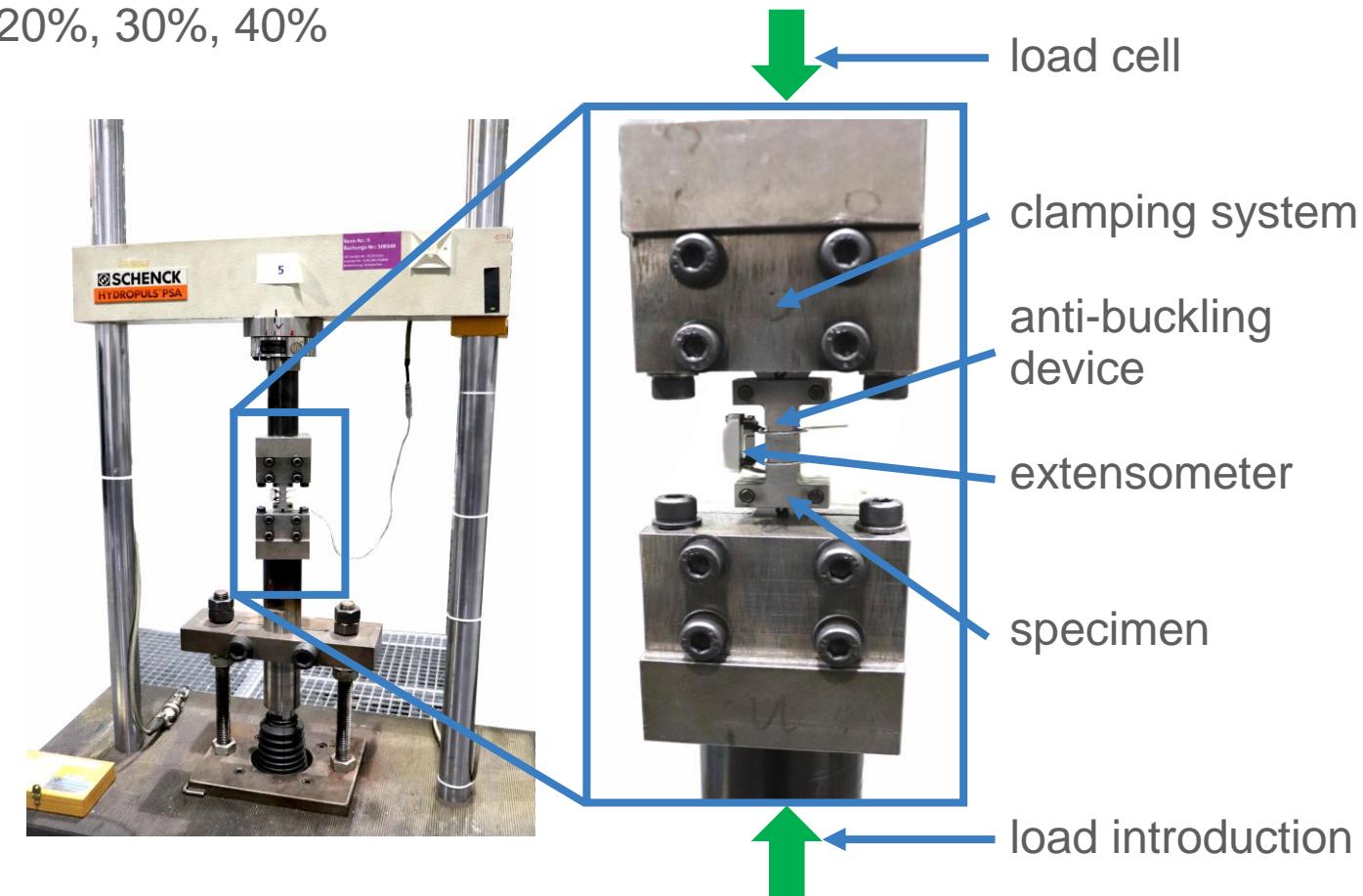
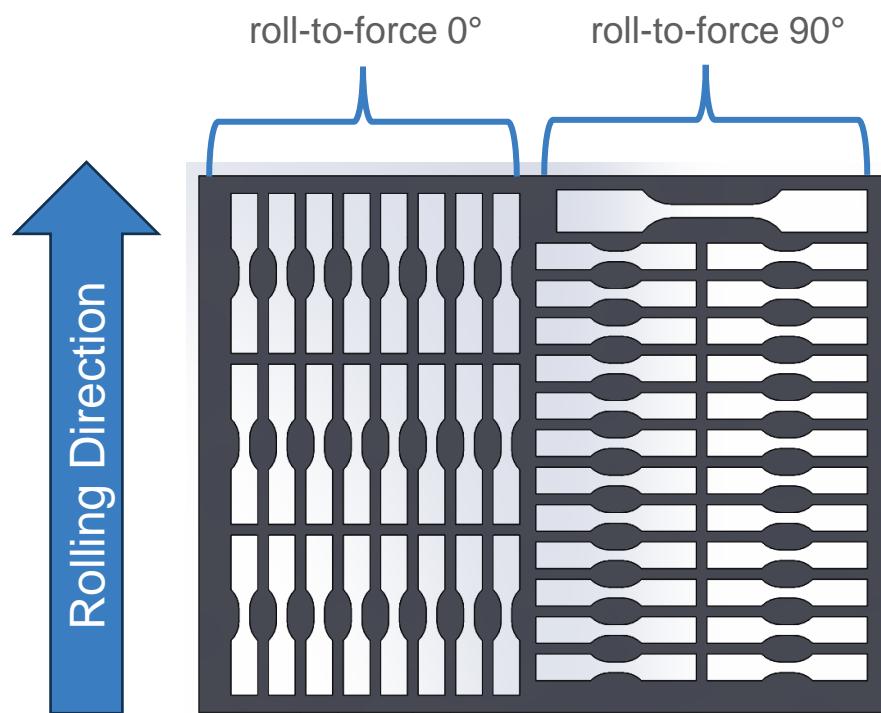


Deriving of material properties for CAE application



Test Campaign

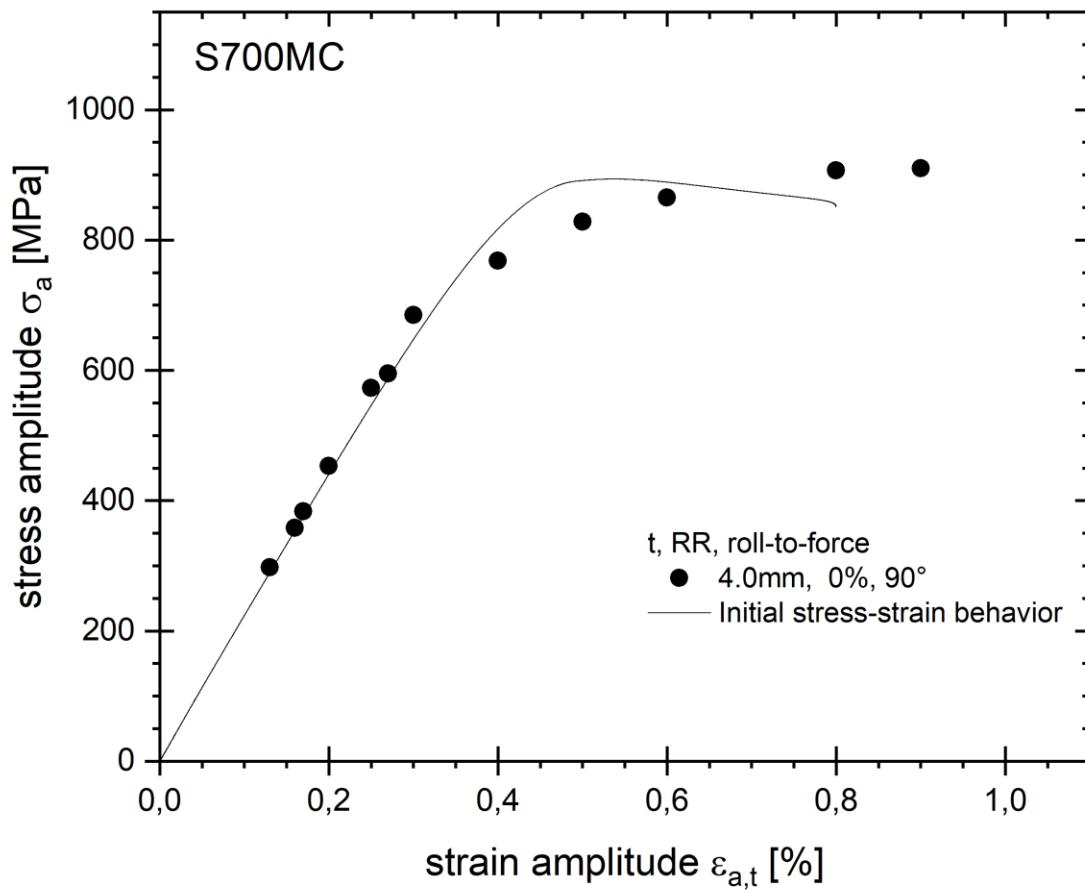
- Strain-controlled fatigue tests [$R=-1$] according to ISO 12106, SEP1240, ASTM E606
- Force-controlled fatigue tests [$R=-1$] according to ISO 12107, DIN 50100
- 5 different rolling reductions (RR): 0%, 10%, 20%, 30%, 40%
- 2 roll-to-force orientations: 0°, 90°



Specimen Test

Stress-Strain Curves

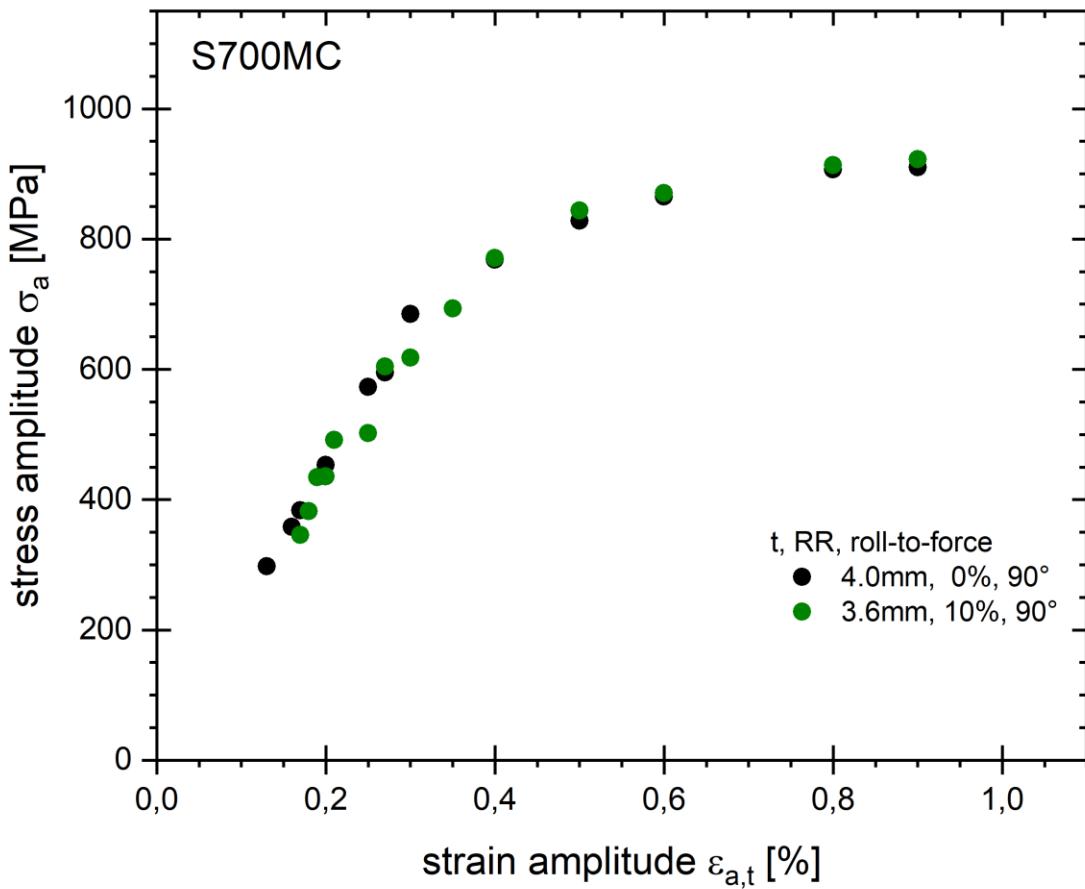
Influence of rolling reduction on the cyclic stress-strain behavior



- Raw material, RR=0%
- Solid line: Initial stress-strain behavior (comparable to quasi-static)
- Black dots: Cyclic stress-strain behavior (focus of investigations in this study)
 - Each datapoint is the result of one individual fatigue test

Stress-Strain Curves

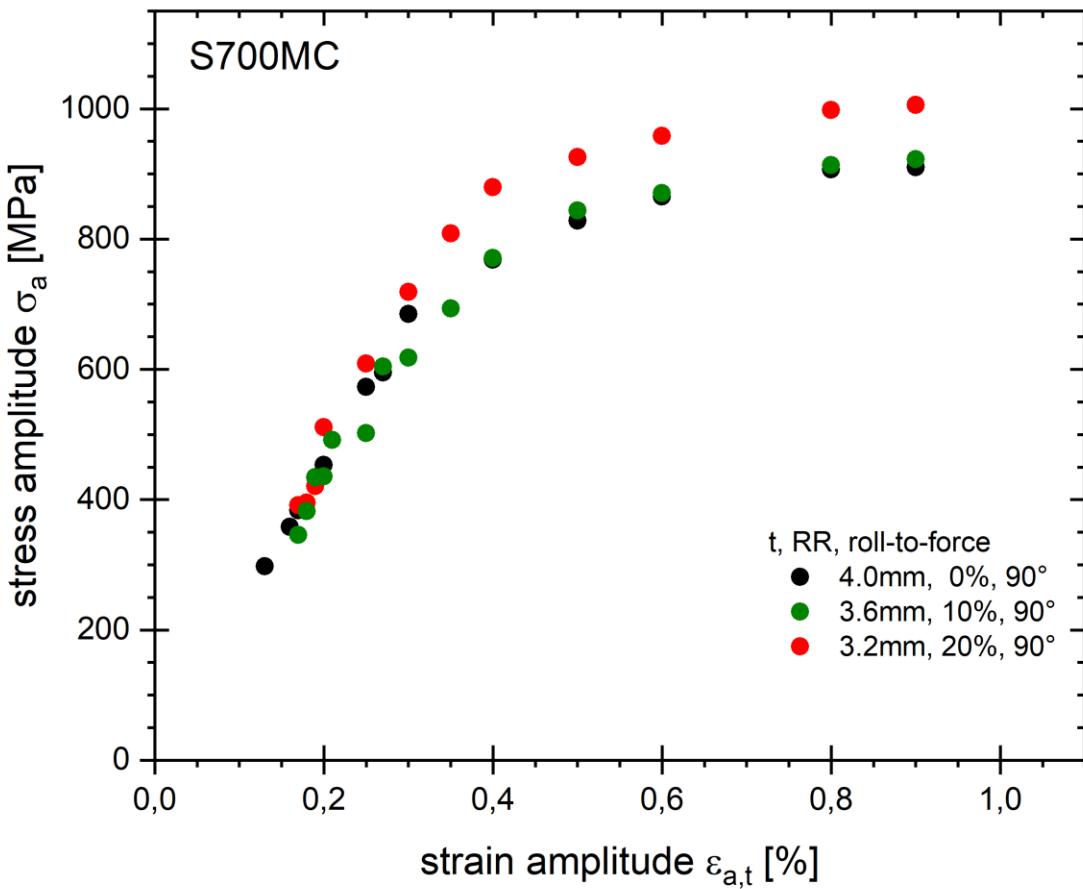
Influence of rolling reduction on the cyclic stress-strain behavior



- RR=10%
- A slight increase in strength with increasing rolling reduction can be observed, yet not very distinctive

Stress-Strain Curves

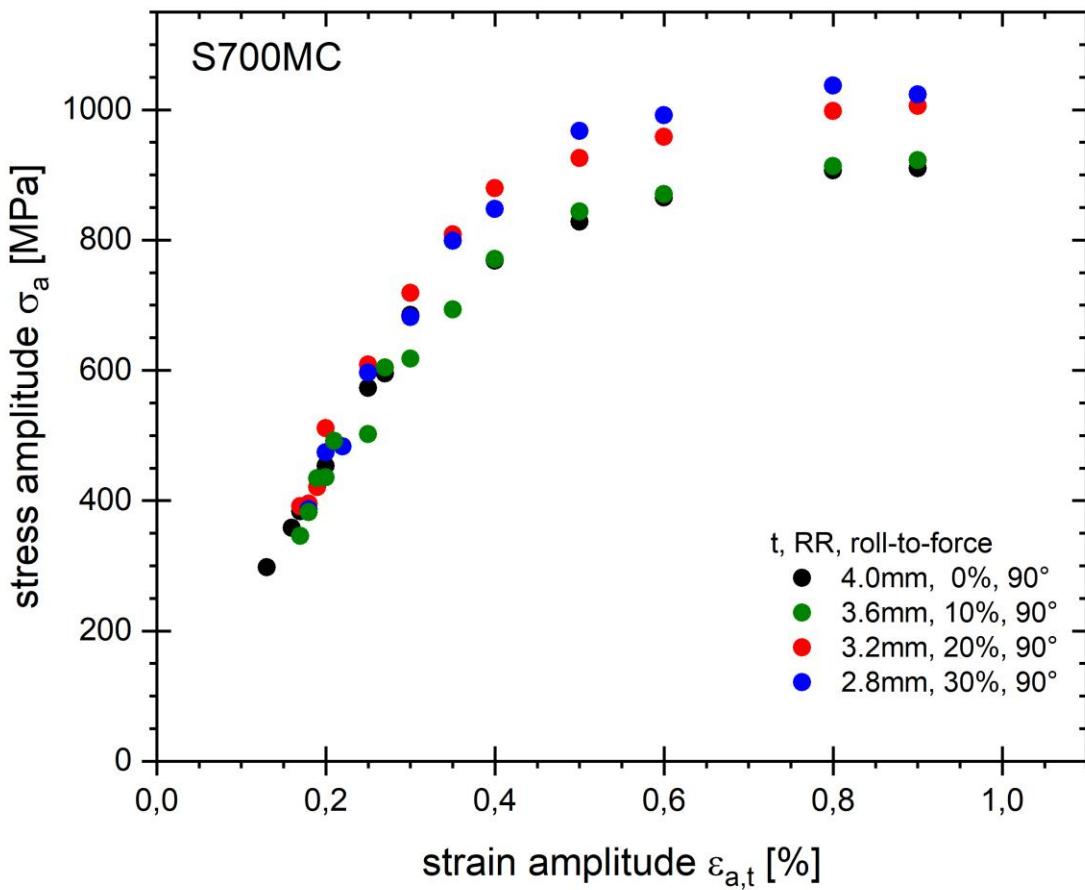
Influence of rolling reduction on the cyclic stress-strain behavior



- RR=20%
- From 20% of rolling reduction onwards the increase in strength becomes apparent
- Increase of around 16% of the stress amplitude at 0.8% of strain compared to initial condition

Stress-Strain Curves

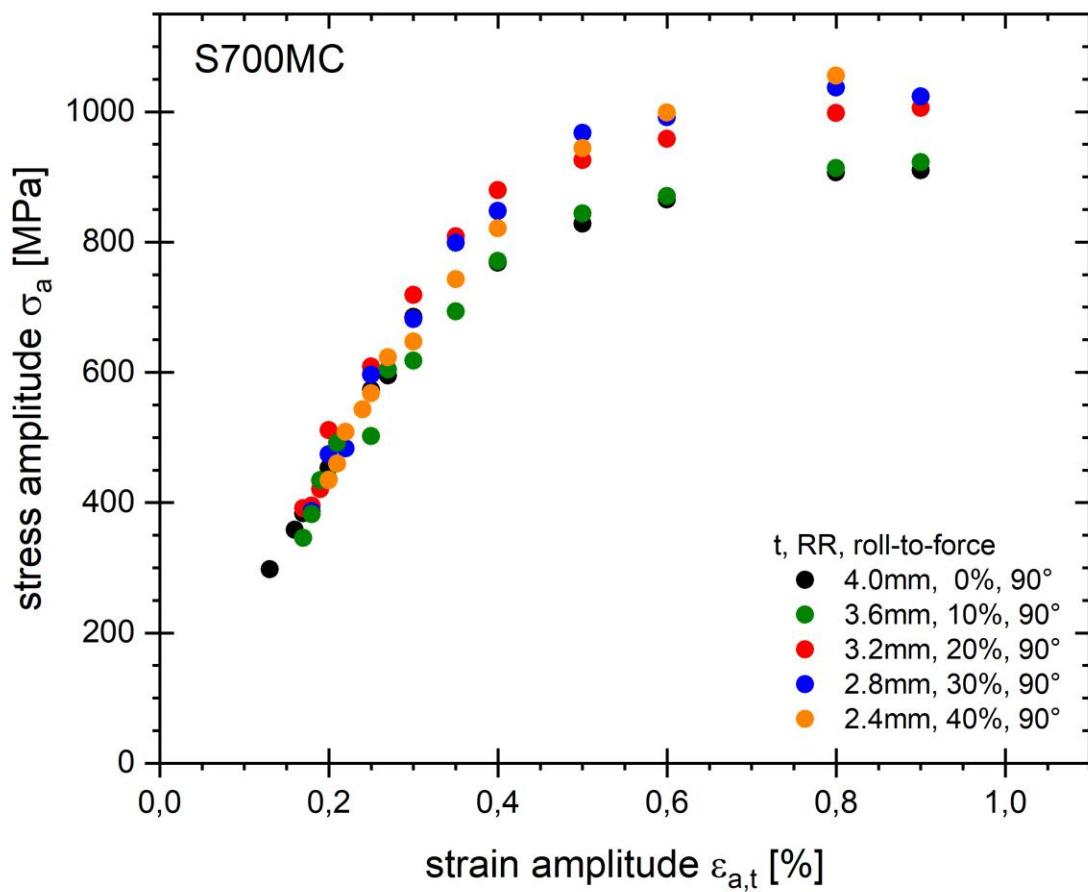
Influence of rolling reduction on the cyclic stress-strain behavior



- RR=30%
- Further increase of strength, but significantly less gain than between 10% and 20%

Stress-Strain Curves

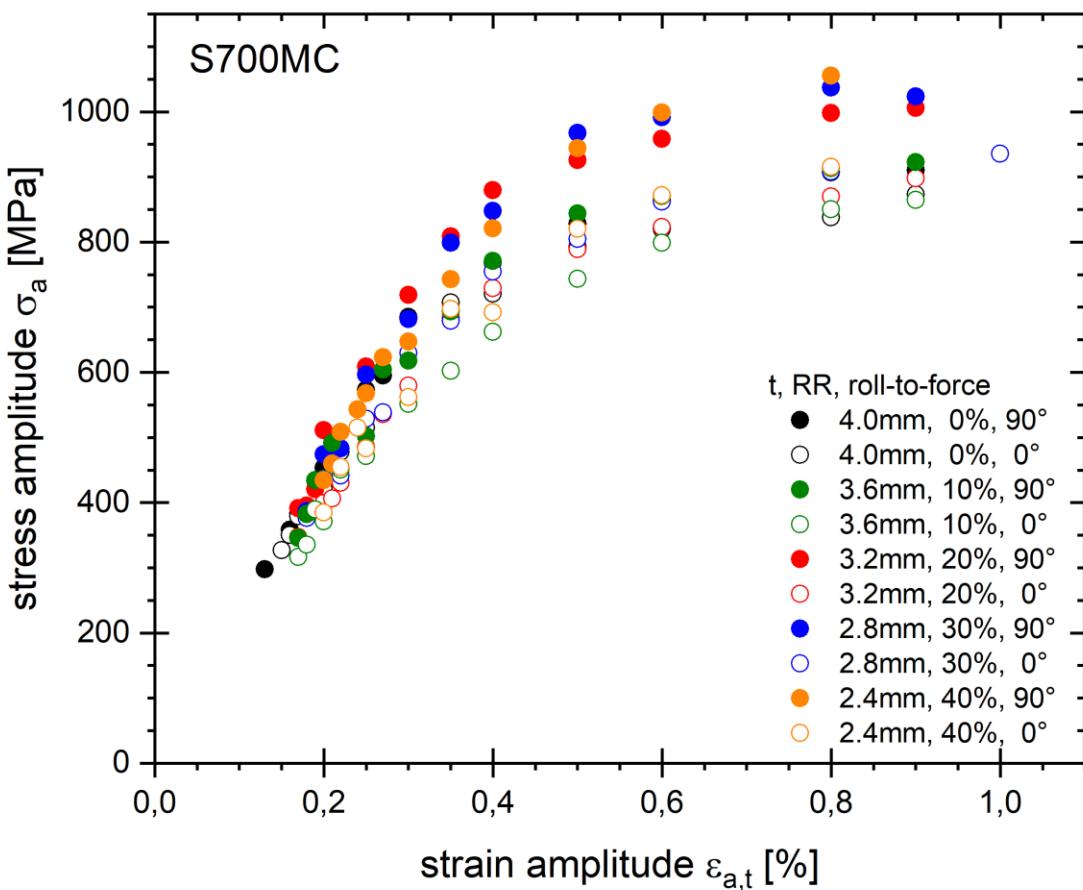
Influence of rolling reduction on the cyclic stress-strain behavior



- RR=40%
- Slight increase of strength

Stress-Strain Curves

Influence of rolling reduction on the cyclic stress-strain behavior



Anisotropic behavior observed due to rolling

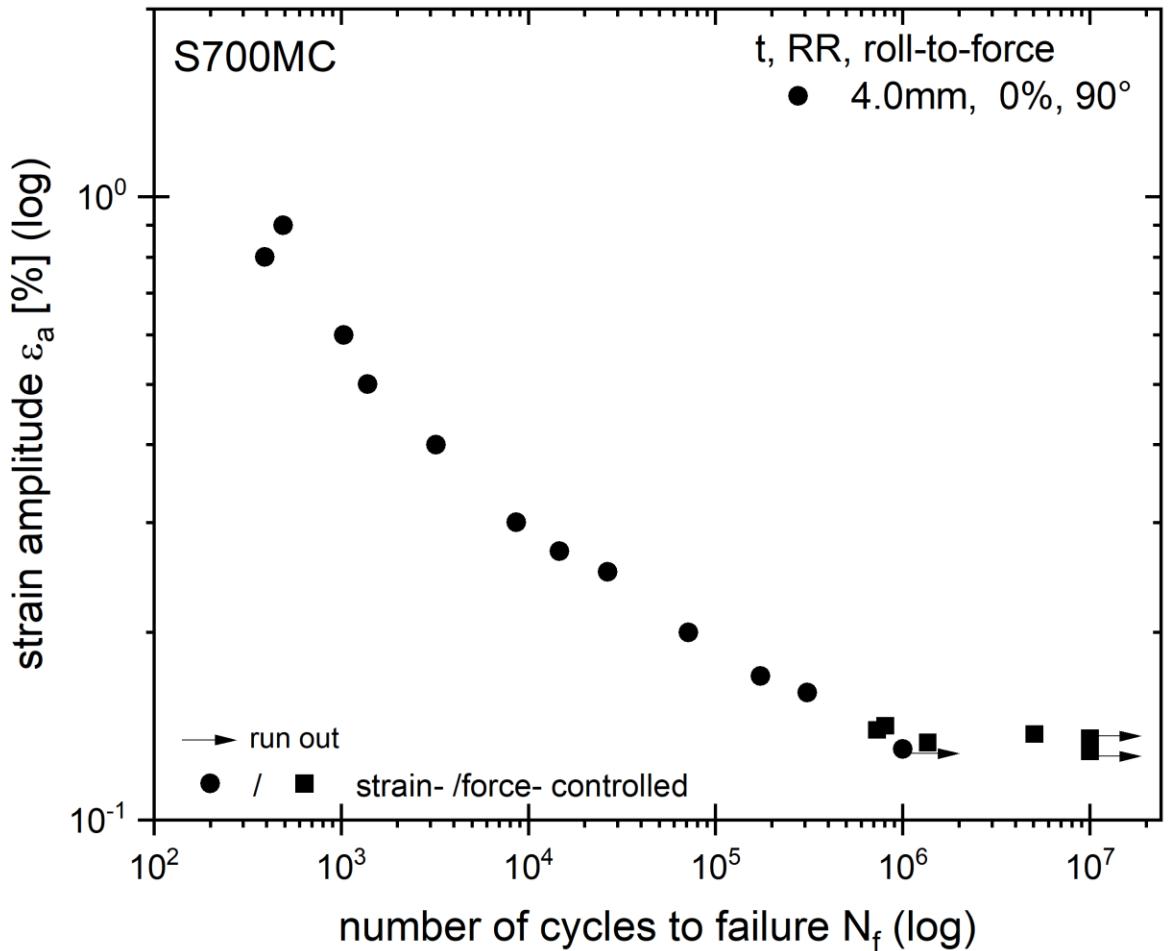
MTH benefit: strength increase can be observed in both directions of roll-to-force

Difference between 0° and 90° for **RR=40%**

- at 0.2% of strain the stress amplitude for the 90° specimen was about **12% higher** than the 0° specimen
- at 0.8% of strain the stress amplitude for the 90° specimen was about **15% higher** than the 0° specimen

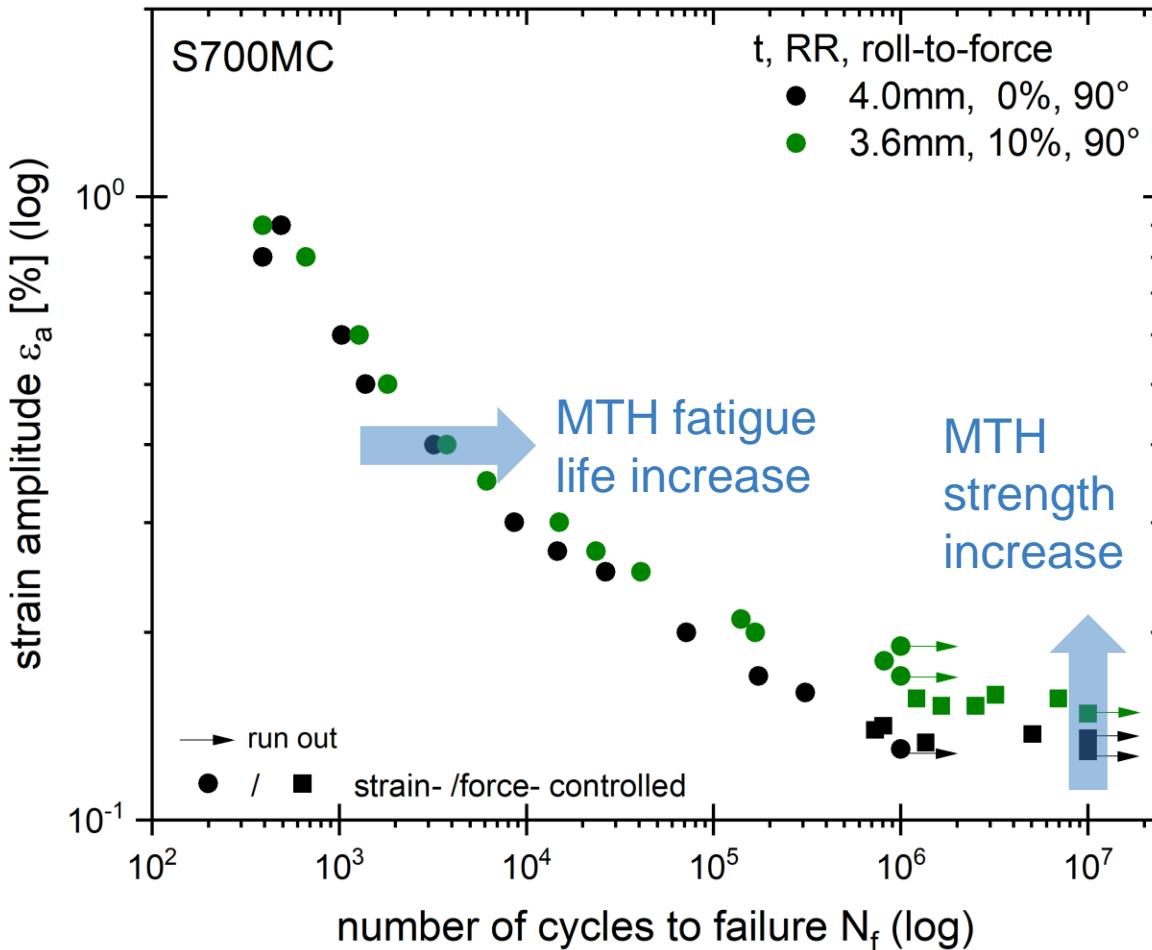
Fatigue Test Results

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- Raw material, RR=0%
- Strain-controlled up to 10^6 cycles
- Force-controlled up to 10^7 cycles

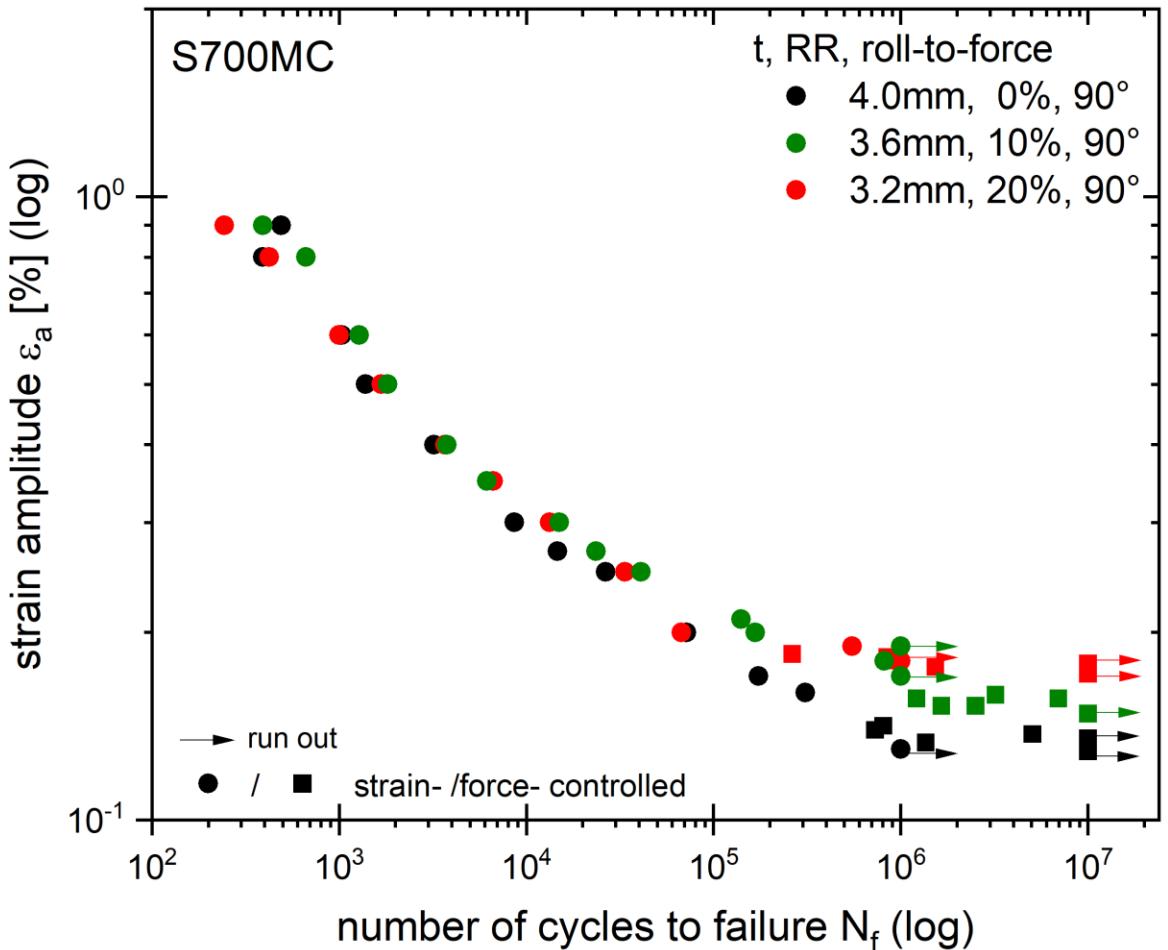
Fatigue Test Results



- RR=10%
- While the influence of 10% rolling reduction was not very distinctive in the stress-strain curve, increase is apparent within the strain-life curve!

Fatigue Test Results

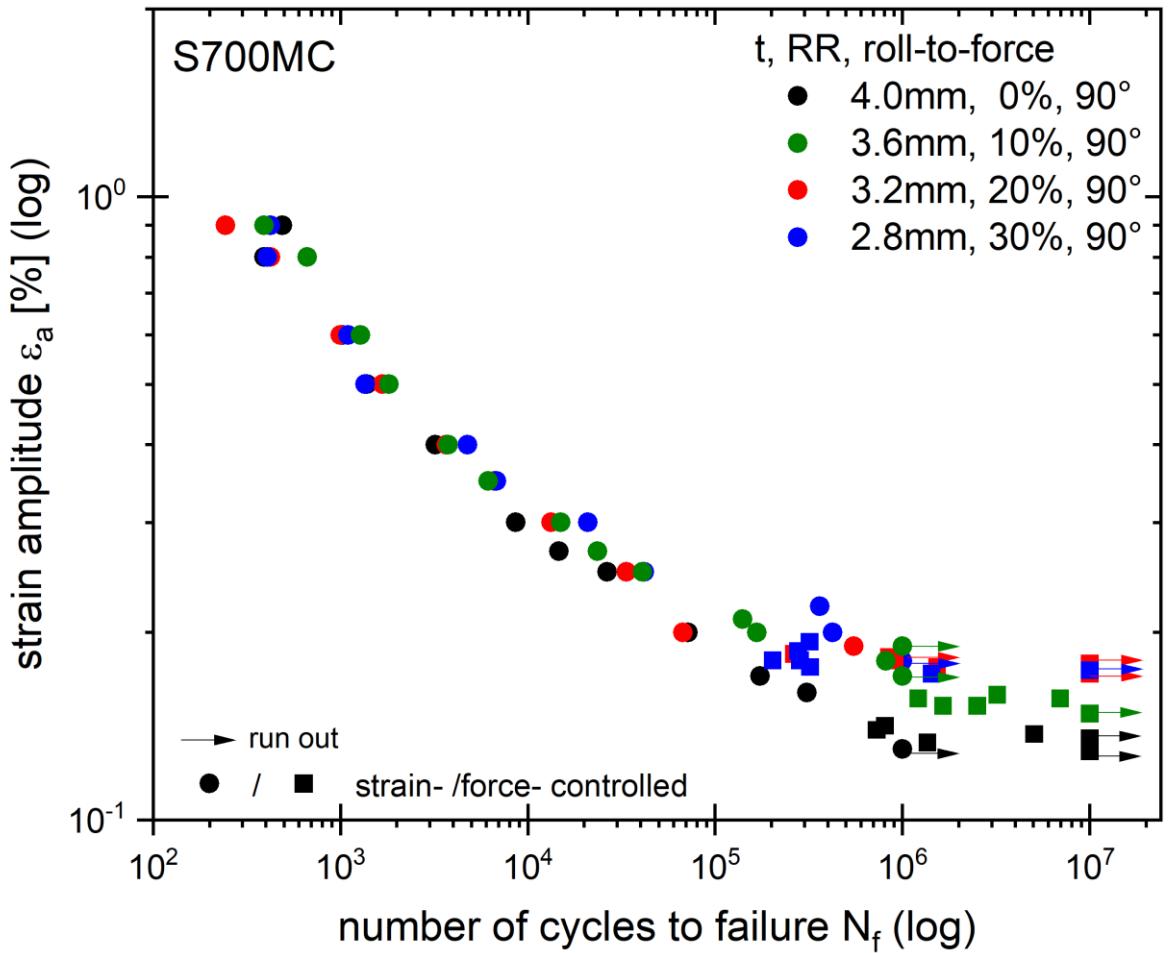
GDIS



- RR=20%
- Further increase in fatigue strength can be seen

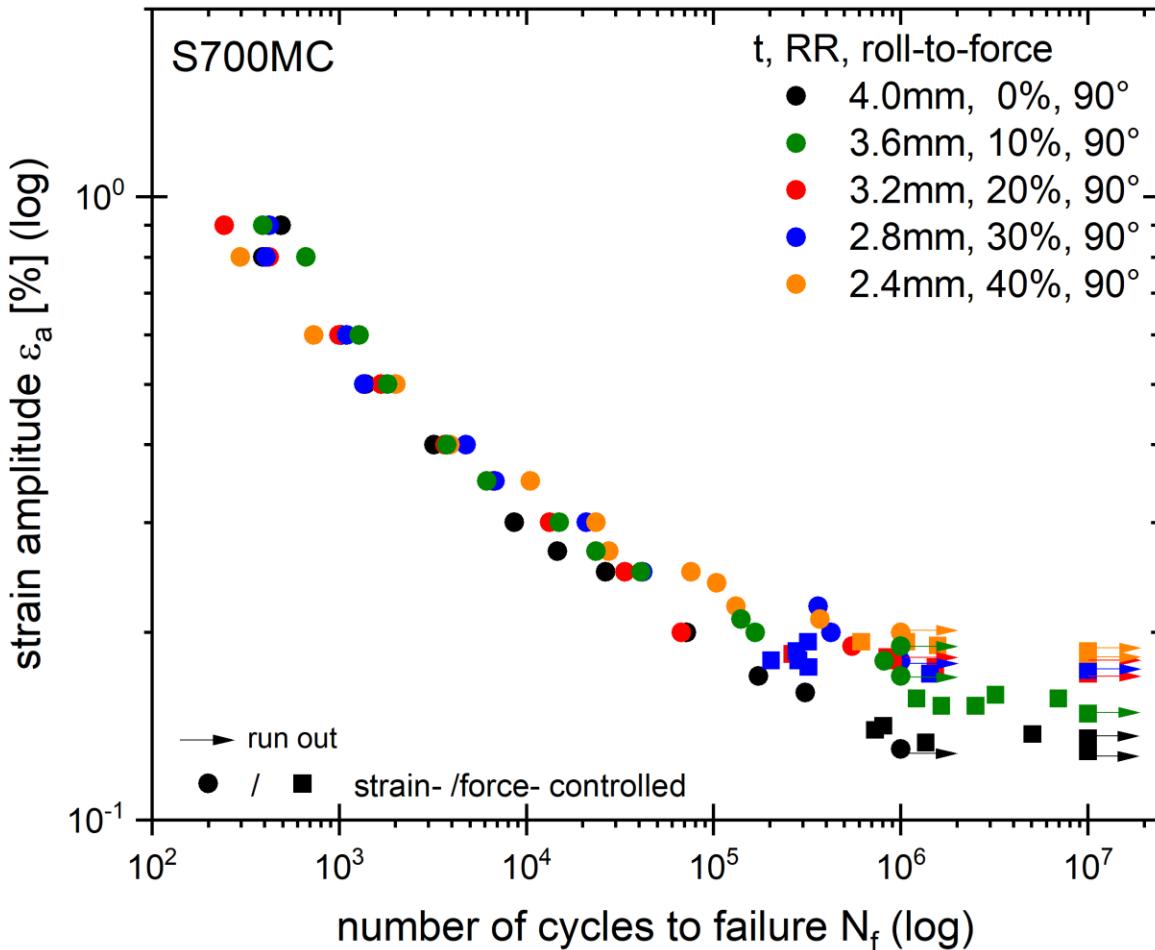
Fatigue Test Results

GDIS



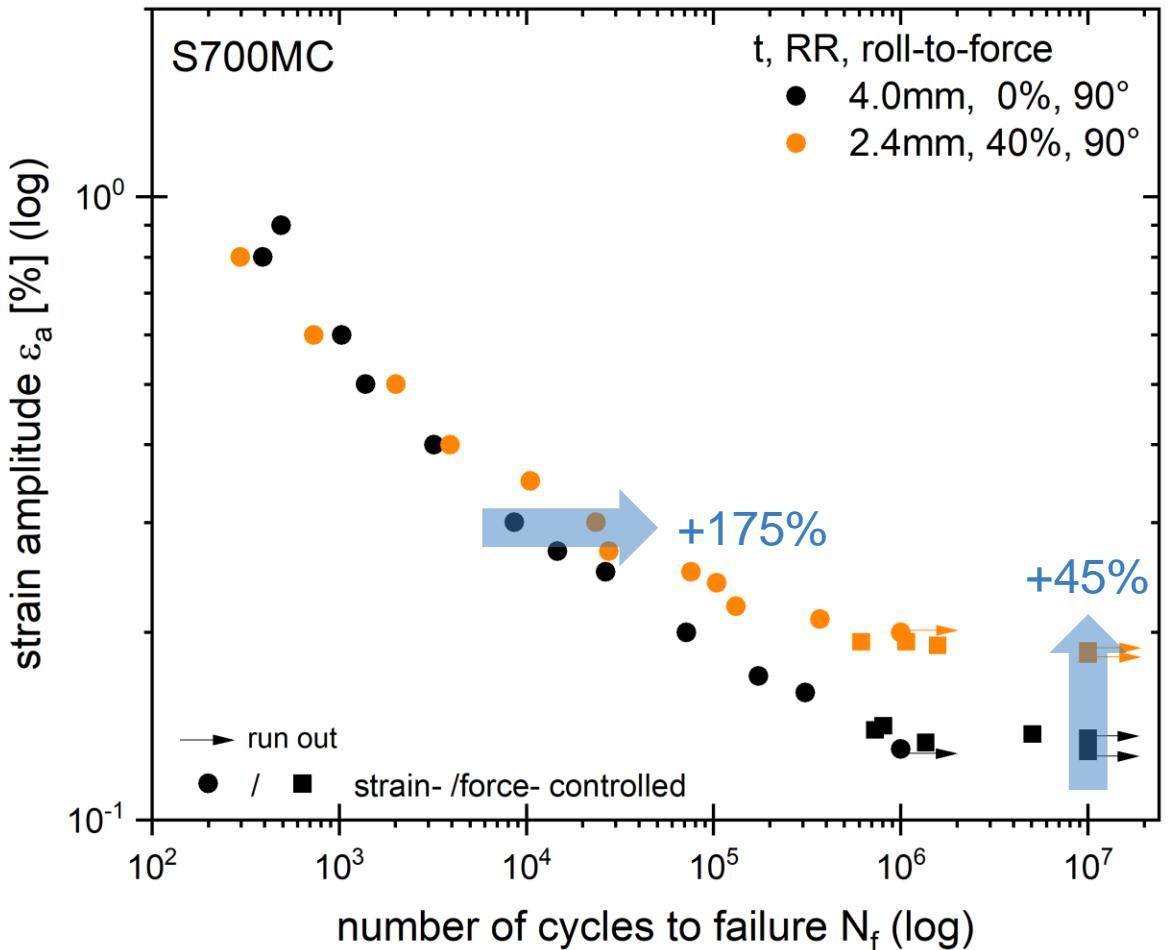
- RR=30%
- Both fatigue life and fatigue strength show significant gains

Fatigue Test Results



- RR=40%
- Further gains
- Fatigue life $>10^6$ for automotive applications:
 - further gains in fatigue strength that can be exploited

Fatigue Test Results

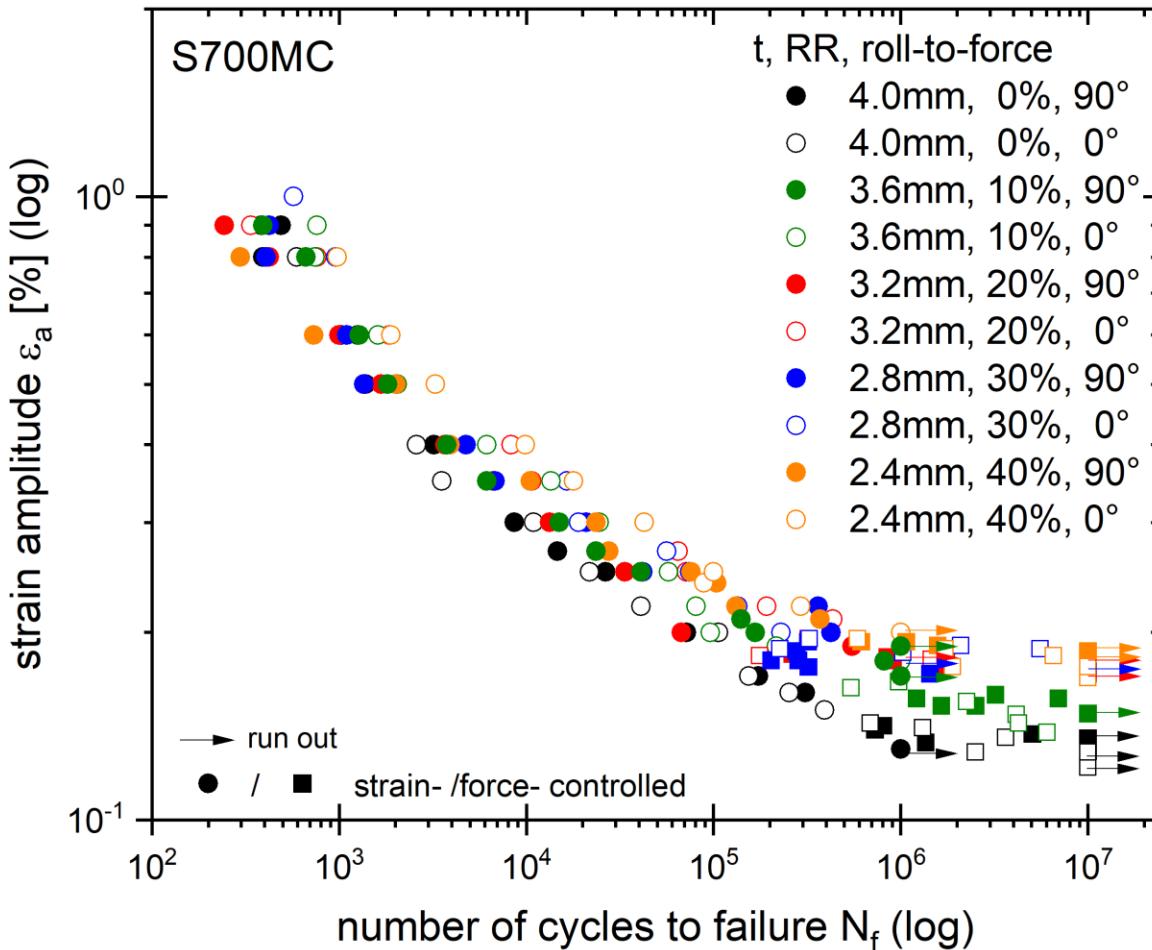


- RR=40%
- Further gains
- Fatigue life $>10^6$ for automotive applications:
 - further gains in fatigue strength that can be exploited
- RR=0 % (Raw material) vs. RR=40%:
 - Fatigue life: $\sim+175\%$ (at $\varepsilon_{a,t}=0.3\%$)
 - Fatigue strength: $\sim+45\%$ (at $N=10^7$)

Fatigue Test Results

GDIS

Anisotropic behavior



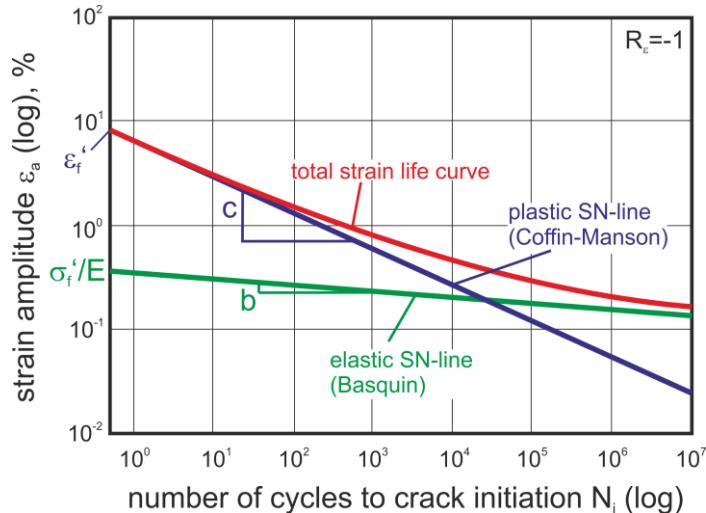
- Comparable behavior for the specimens extracted at 0° roll-to-force

Fatigue Behavior – Math. Approach (CAE-Application)

Cyclic Material Behavior

Strain-life and cyclic stress-strain curves

Basquin-Coffin-Manson-Morrow



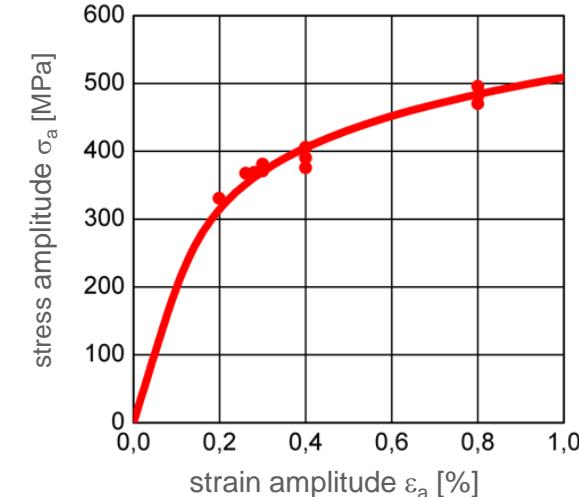
$$\varepsilon_{a,t} = \varepsilon_{a,e} + \varepsilon_{a,p} = \frac{\sigma_f'}{E} \cdot (2N)^b + \varepsilon_f' \cdot (2N)^c$$

σ_f'	fatigue strength coefficient
b	fatigue strength exponent
ε_f'	fatigue ductility coefficient
c	fatigue ductility exponent

$$n' = \frac{b}{c}$$

$$K' = \frac{\sigma_f'}{\varepsilon_f'^{-\frac{b}{c}}}$$

Ramberg-Osgood

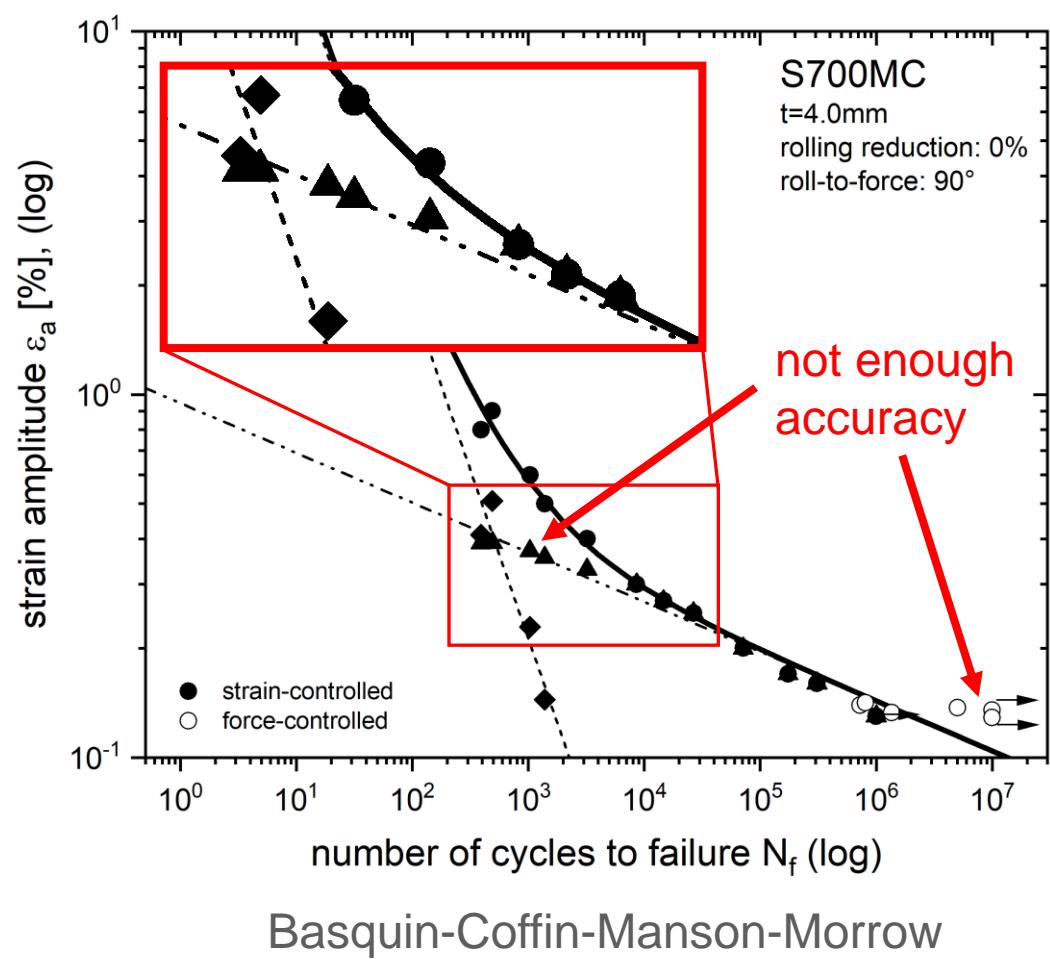
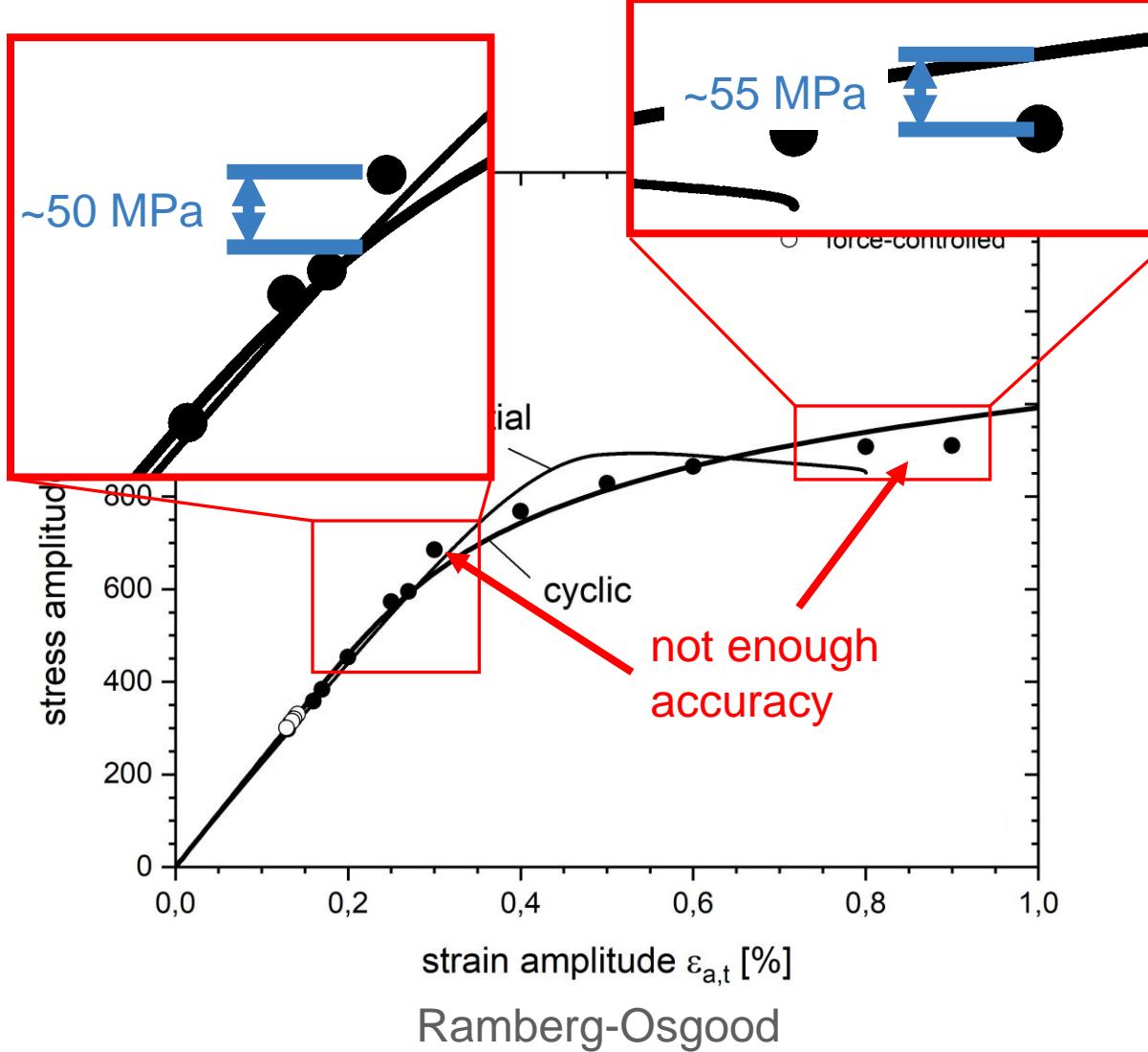


$$\varepsilon_{a,t} = \varepsilon_{a,e} + \varepsilon_{a,p} = \left(\frac{\sigma_a}{E} \right) + \left(\frac{\sigma_a}{K'} \right)^{\frac{1}{n'}}$$

K'	cyclic hardening coefficient
n'	cyclic hardening exponent

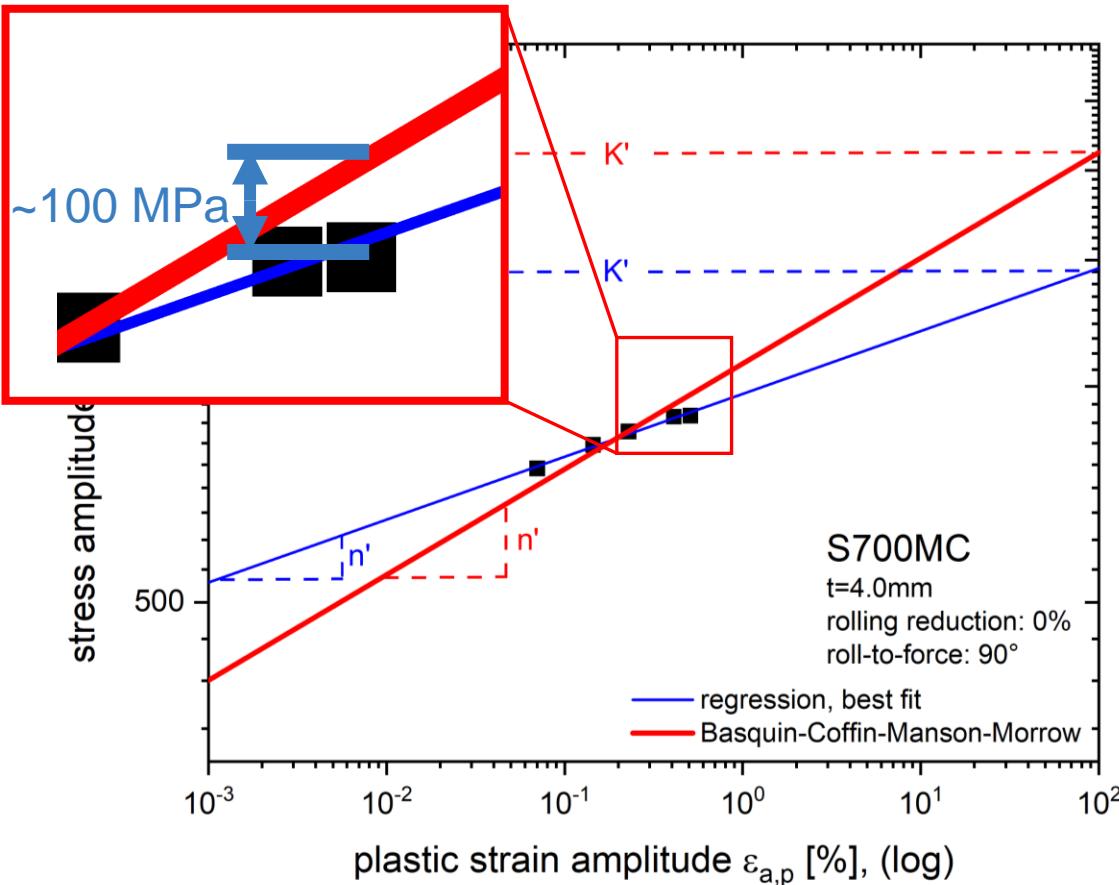
Fatigue Behavior – Math. Approach (CAE-Application)

Cyclic stress-strain and strain-life curve / Conventional models



Check of the material model

Cyclic flow curve – $\log(\sigma_a)$ vs. $\log(\varepsilon_{a,p})$ -curve – compatibility condition



Red line:

- Determined using the compatibility condition
- Basquin-Coffin-Manson-Morrow does not have enough accuracy

Blue line:

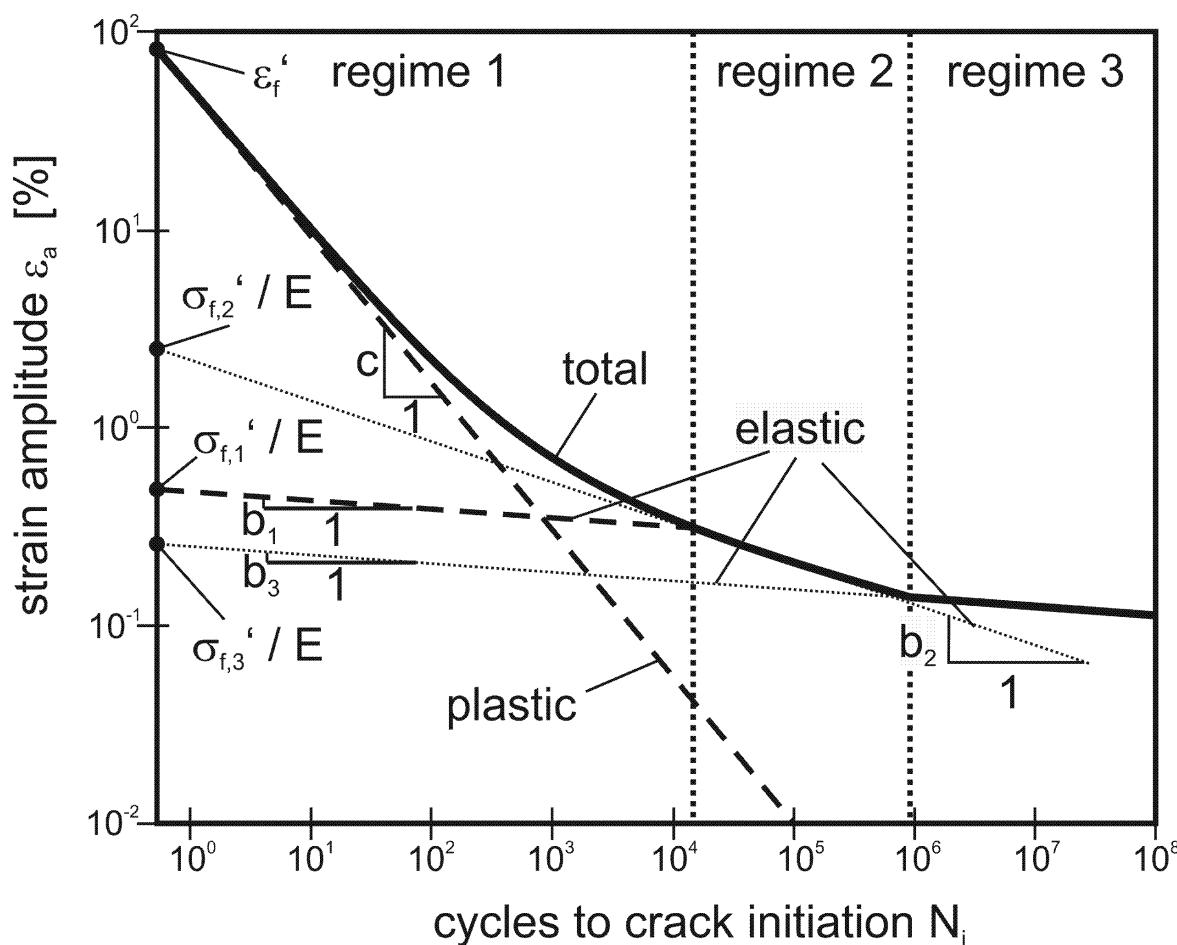
- Fits the datapoints according to regression
- Significantly different values K' and n' for the cyclic stress-strain curve according to Ramberg-Osgood

$$\begin{aligned} R^2_{BCMM} &= 0.521 \\ R^2_{REGRESSION} &= 0.985 \end{aligned}$$

Fatigue Life Curve – New Math. Approach

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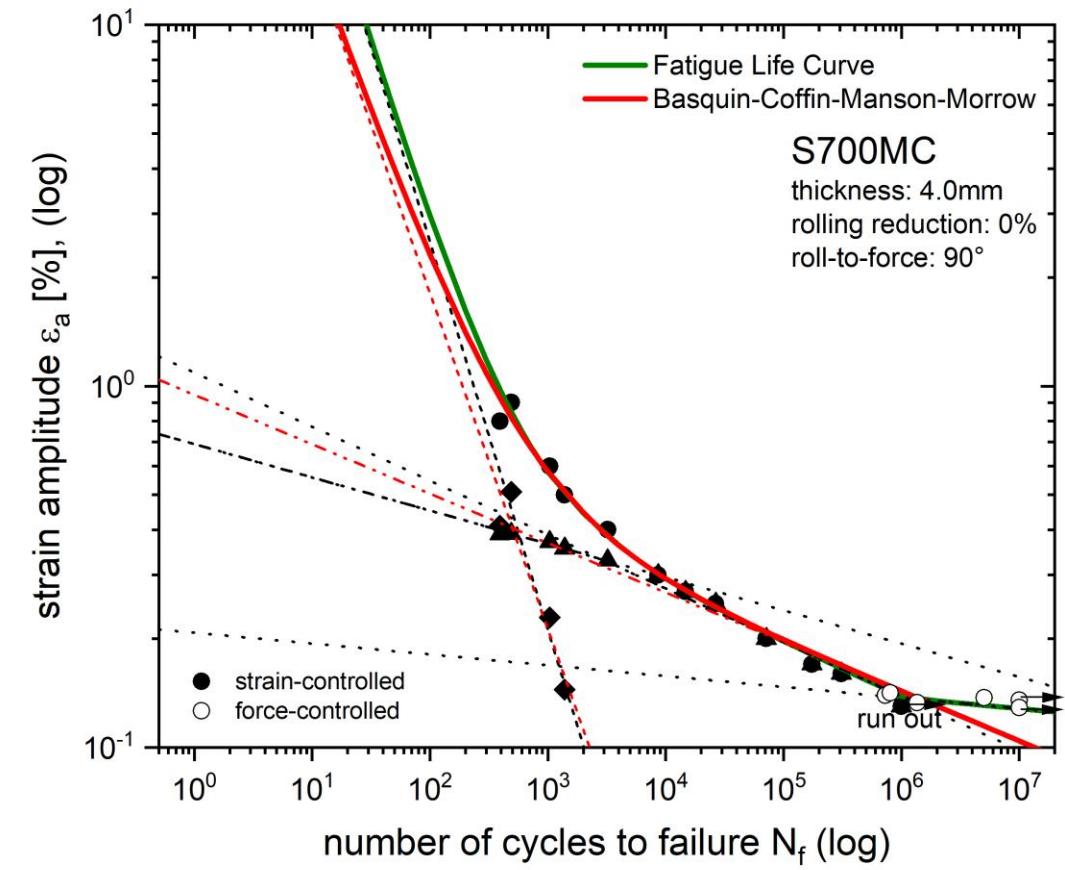
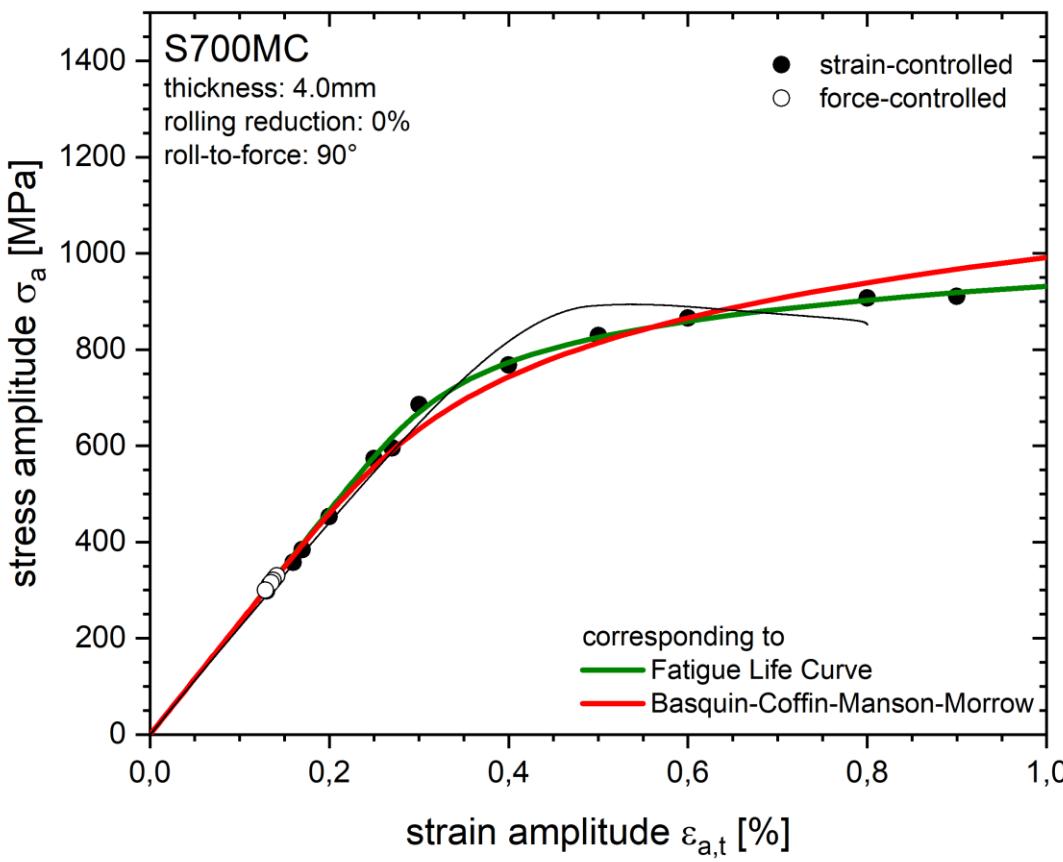
Characteristic values



- Fatigue Life Curve as an improved approach with higher accuracy to describe cyclic material behavior compared to conventional models
- Partition of the strain-life curve into three regimes depending on the cyclic stress-strain behavior
 - Regime 1: elastic-plastic
 - Regime 2: transition zone
 - Regime 3: macroscopic elastic
- Combination of strain- and force-controlled test results possible with Fatigue Life Curve
- Recommended by German Association of the Automotive Industry (VDA) 239:300

(Source: R. Wagener und T. Melz, "Deriving a continuous fatigue life curve from LCF to VHCF", SAE Technical Paper, 2017-01-0330, 2017.)

Fatigue Life Curve

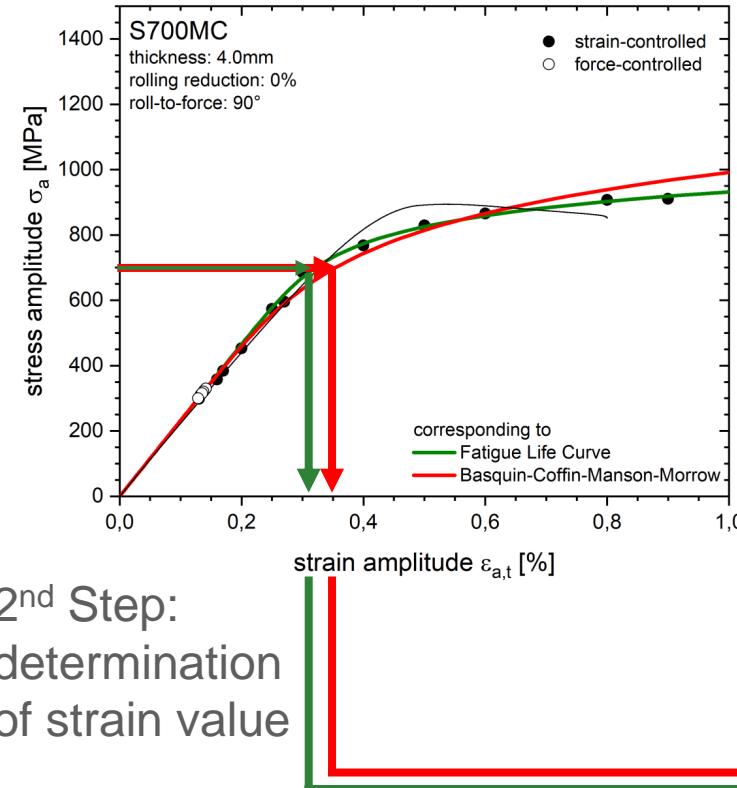


Deriving of Calculated Fatigue Life

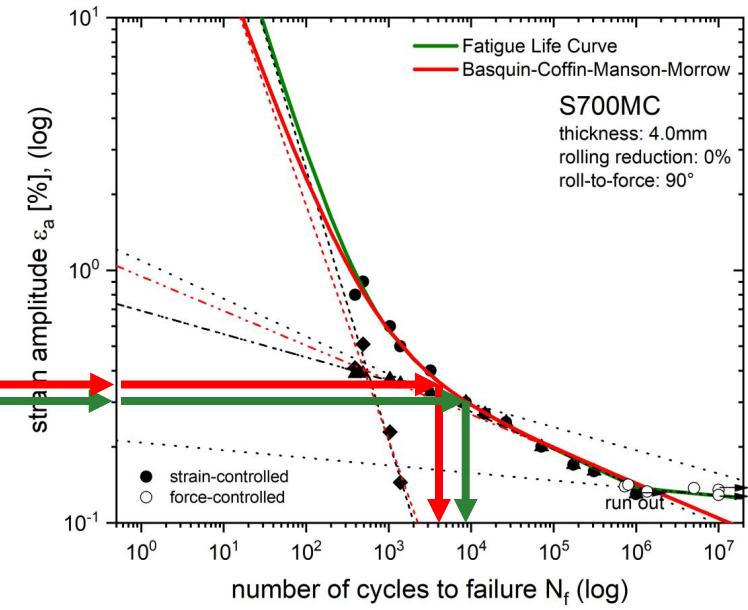
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1st Step: stress calculation of applied force



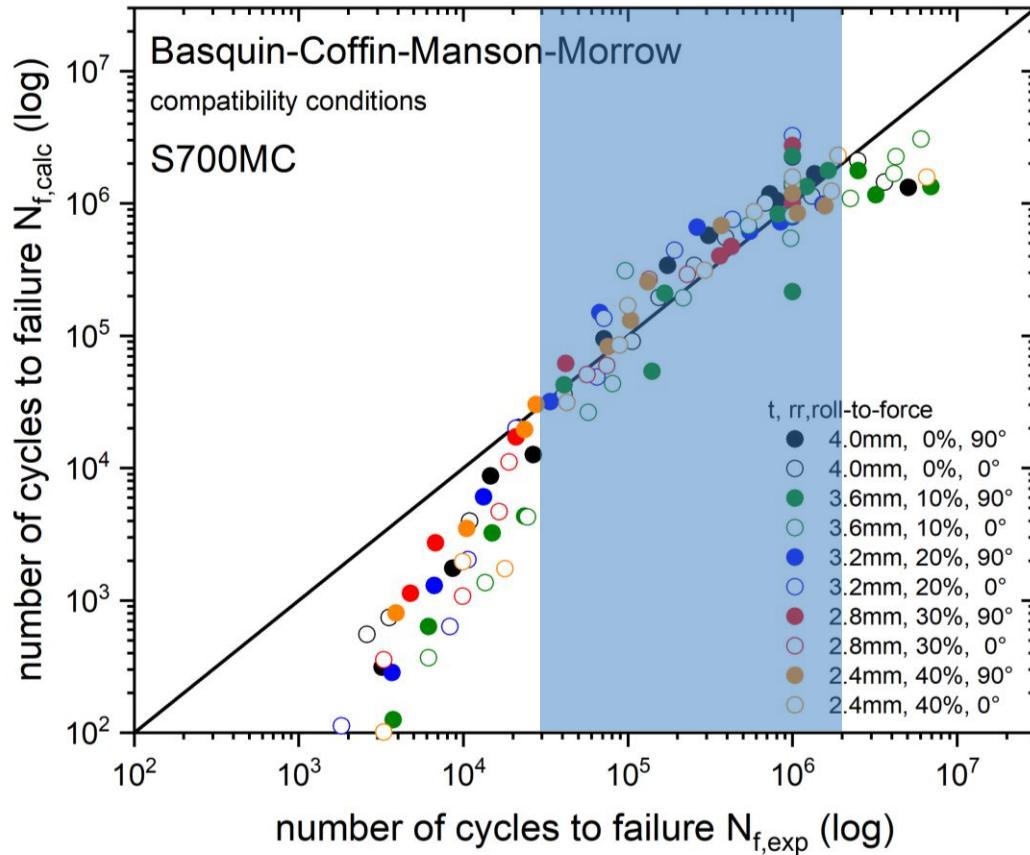
2nd Step:
determination
of strain value



3rd Step: calculation of fatigue life

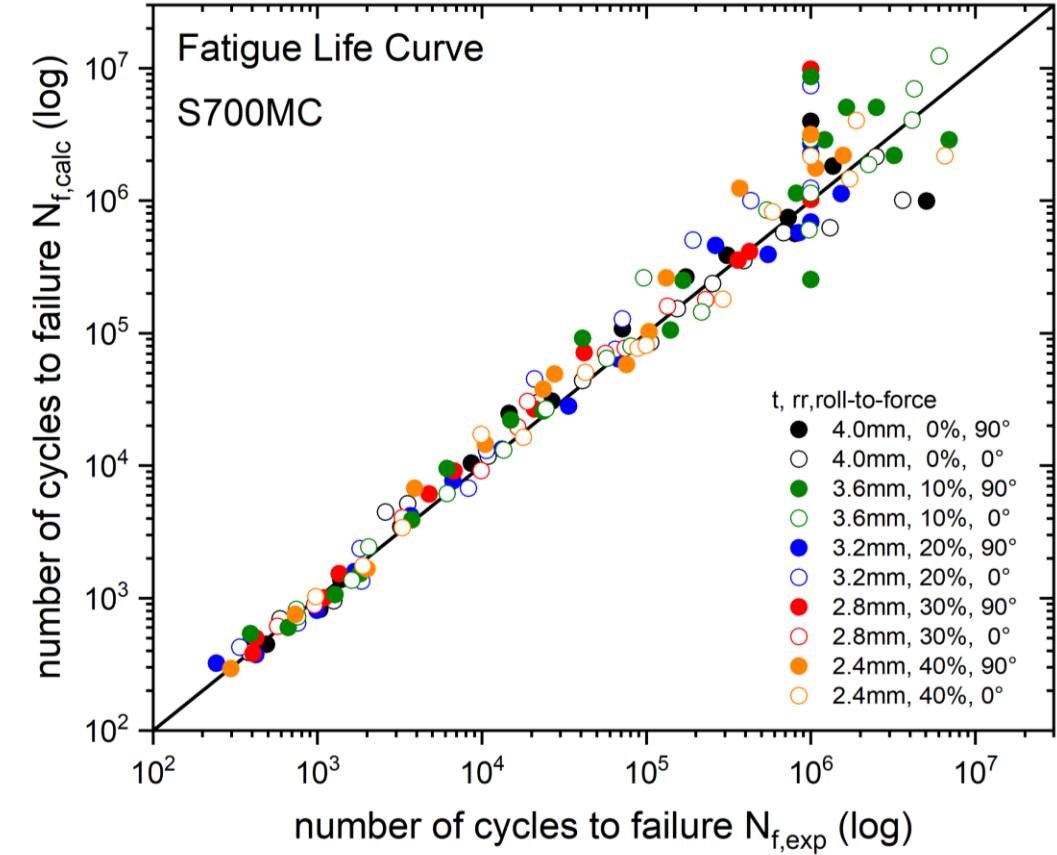
Comparison of Calculated and Experimental Fatigue Life

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$$R^2_{BCMM(\text{overall})} = 0.754$$

$$R^2_{BCMM(30,000 > x > 2,000,000)} = 0.801$$

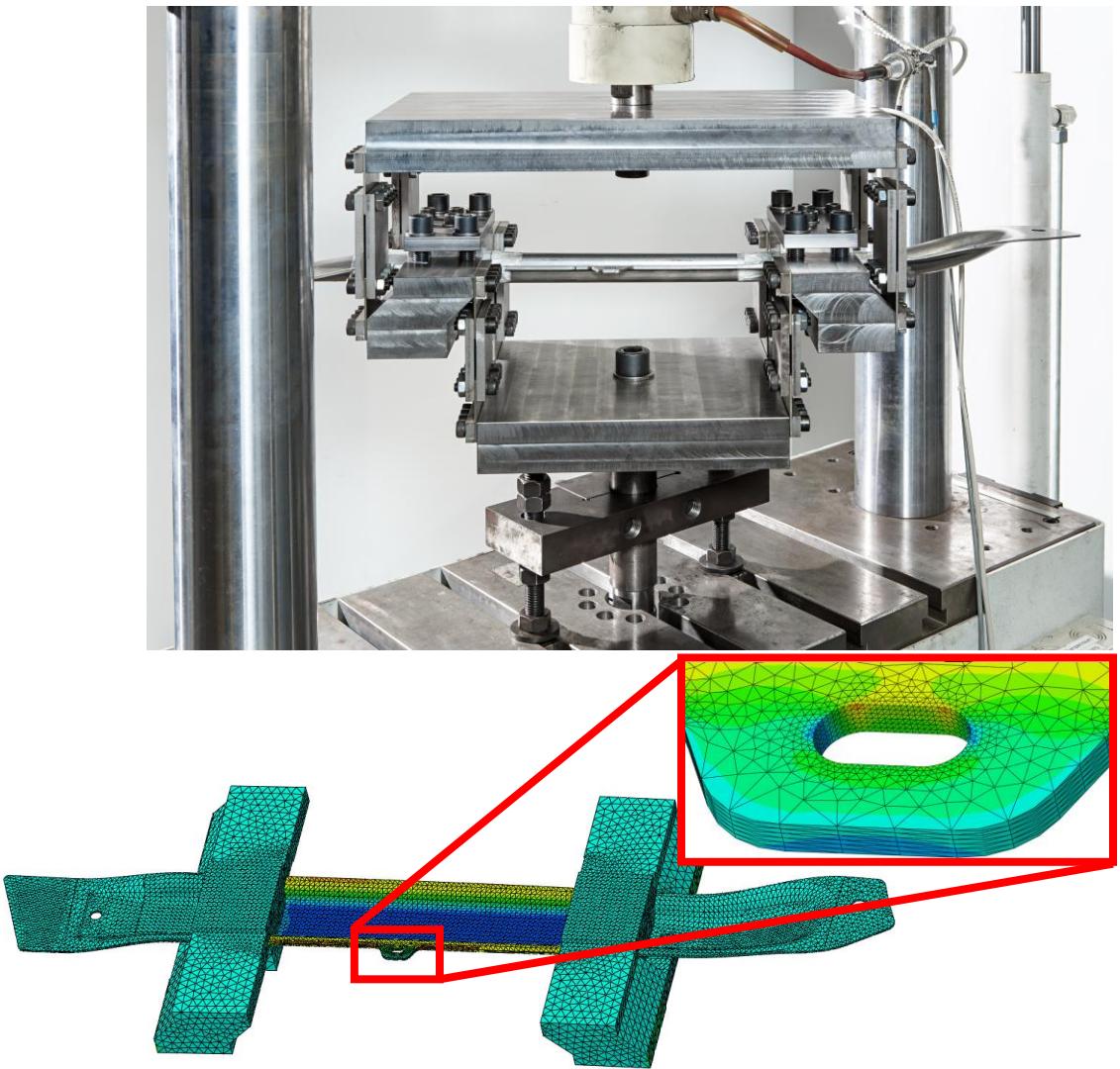
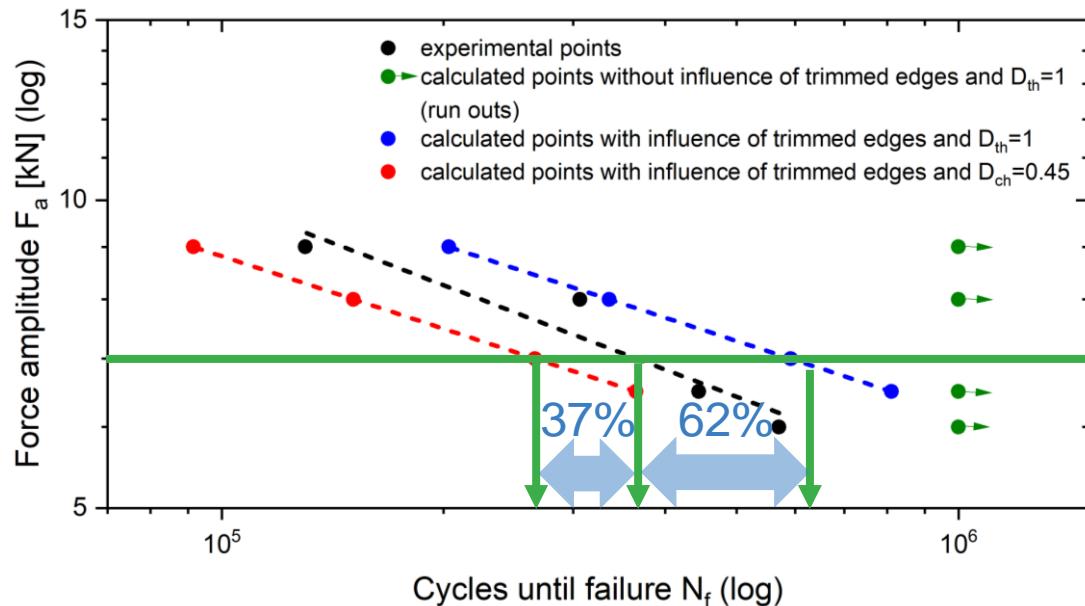


$$R^2_{FLC} = 0.932$$

Part Validation

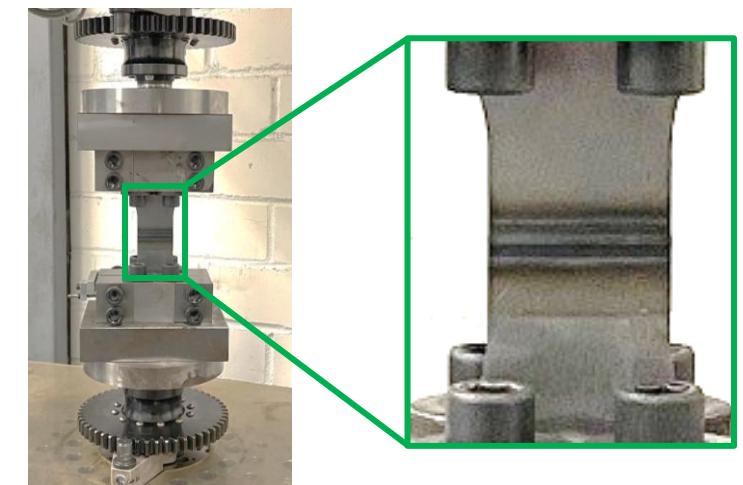
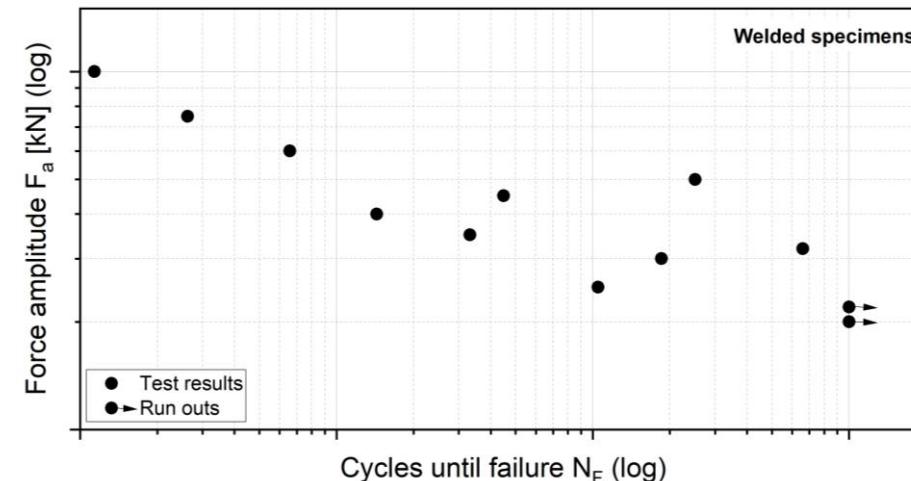
Part Validation

1. Strains calculated from FE
2. Taking into account influence of edges by shifting the Fatigue Life Curve downwards through multiplication of strain value at $N=1000$ ($\times 0.6$) and $N=100000$ ($\times 0.73$) [1]
3. Fatigue Life from Fatigue Life Curve using FE-strains
4. Using $D_{\text{characteristic}}$ instead of $D_{\text{theoretical}}$ takes into account force-controlled bending loading [2]
5. Comparison calculated fatigue life vs. experimental data



Summary & Outlook

- MTH has a positive influence on the structural durability by increasing rolling reduction leading to:
 - Increase in fatigue life (reduced strain amplitude at same stress level)
 - Increase in fatigue strength (higher stress level for same strain amplitude)
- The Fatigue-Life Curve provides much more accuracy compared to the traditional approaches (Basquin-Coffin-Manson-Morrow and Ramberg-Osgood)
- Validation (Fatigue-Life Curve vs. experimental results) using automotive part shows satisfactory correlation
- Next step: Weld seams



Thank you for your attention!

GDIS



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